

تقدّم لجنة ElCoM الأكاديمية

cafتر لماقة:

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جزيل الشكر للطالبة:

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## Ch.1 8 Introduction

### # Modeling Engineering System Dynamics →

The modeling process of engineering system dynamics starts by identifying the fundamental properties of an actual system.

The minimum set of variables necessary to fully define the system configuration is formed of the degrees of freedom (DOF).

It is then necessary to utilize an appropriate modeling procedure that will result in the mathematical model of the system.

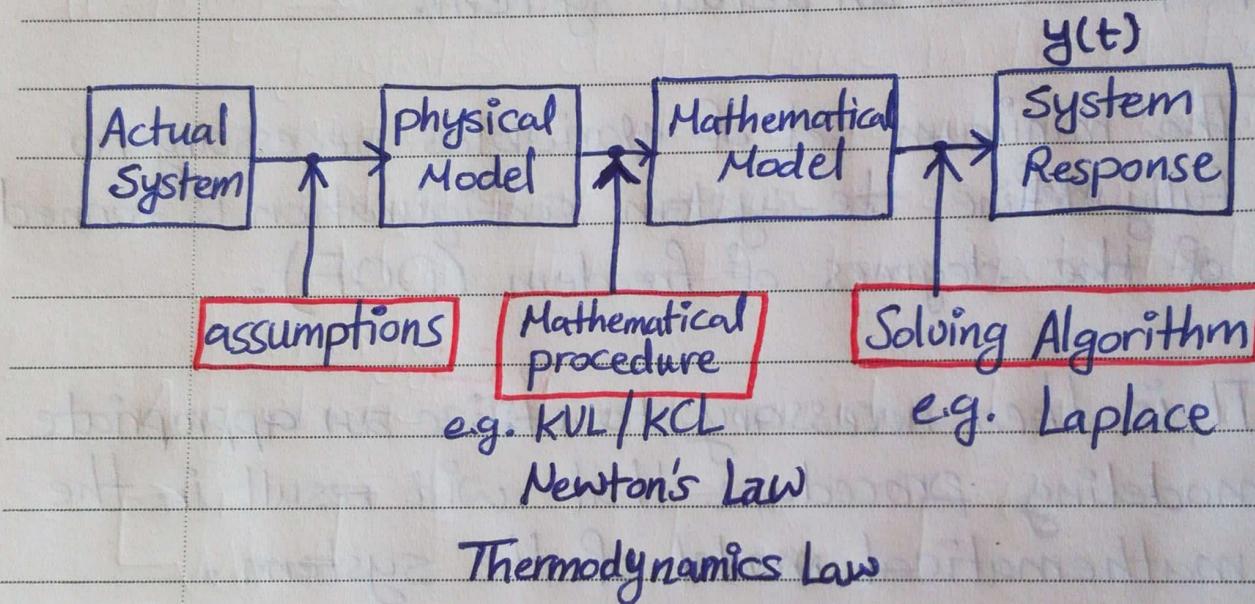
Generally ..

A mathematical model describing the dynamic behaviour of an engineering system consists of a differential equation combining parameters with known/unknown functions and derivatives.

Then, Solving the mathematical model through

adequate mathematical procedures that deliver the solution.

∴ Mathematical Model  $\rightarrow$  Equations that describes the relationship between the inputs and the outputs of the system.

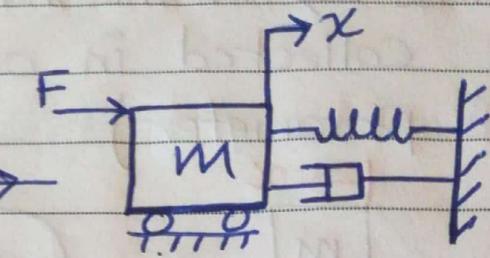
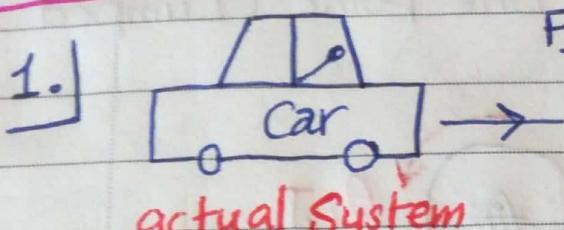


the fig. above shows the flow in a process connecting an actual dynamic system to its response.

m لیسا گلر لیسا مولی نیز یک دلیل است  
.. لیسا مولی نیز دلیل است

## Examples

1.]



physical Model

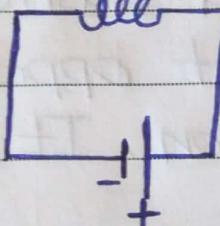
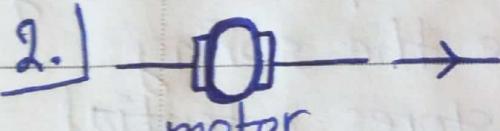
$$\sum F = m \ddot{x} \rightarrow x(t) =$$

$$\dot{x}(t) =$$

$$\ddot{x}(t) =$$

مقدار زمانی تغییرات

2.]



$$KVL / KCL \rightarrow I(t) =$$

$$U(t) =$$

\* Types of Systems  $\Rightarrow$

□ Mechanical System

Basic Elements : a. mass b. Spring c. damper

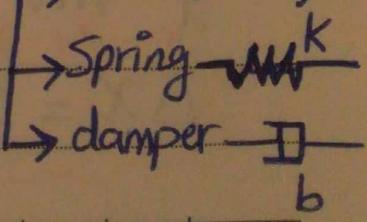
Input : a. Force or b. Torque

Output : a. Displacement b. velocity  
c. acceleration

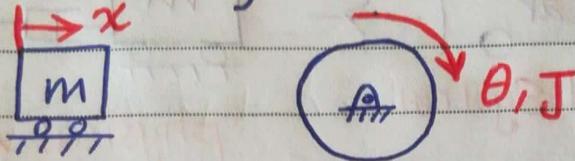
Types of Motion : a. Translational  $\rightarrow$  mass (m)

b. Rotational  $\rightarrow$  Inertia (J)

$\rightarrow$  Rotational spring  $\rightarrow$  Rotational damper

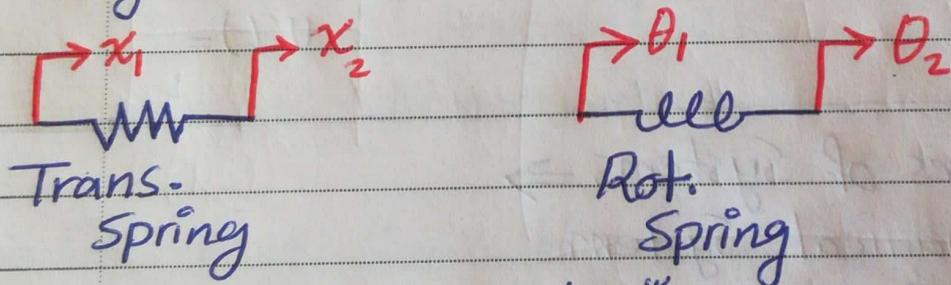


# mass  $\rightarrow$  all the mass of any object is collected in one point (Lumped Parameter)

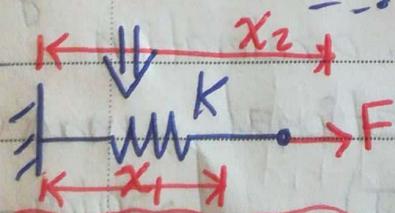


# Spring  $\rightarrow$  Mechanical element, "elastic", that generates elastic force in translatory motion and elastic torque in rotary motion that oppose the spring deformation. It stores potential energy.

Any elastic action is considered as a spring.



لهم اسْعِنْ طَوَّافَةَ سَيْرَنْ  
بِمَ افْلَأْتُكَ مِنْ لَعْنَةِ حَمَّارِي  
يَوْمَ الْجِيْهَنْ



$$\Delta x = x_2 - x_1$$

$$F_s = k \cdot \Delta x = k(x_2 - x_1)$$

Spring Force

For Rot. Spring  $F_s = k \cdot \Delta\theta = k(\theta_2 - \theta_1)$

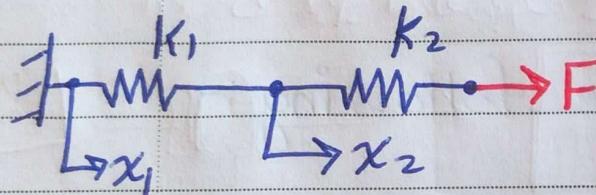
$$E_p = \frac{1}{2} k (x_2 - x_1)^2 = \frac{1}{2} k (\theta_2 - \theta_1)^2$$

„Potential Energy“

\* Combination of Springs 80

a. Series  $\rightarrow$

$$F = kx$$



$$F_{s1} = F_{s2}$$

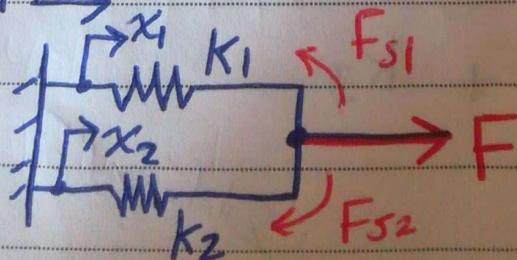
$$x_{eq} = x_1 + x_2$$

$$\frac{F}{k_{eq}} = \frac{F_{s1}}{k_1} + \frac{F_{s2}}{k_2} \Rightarrow k_{eq} = \frac{k_1 k_2}{k_1 + k_2}$$

الجهة التي تبدي المفعول هي k2 ولكن k1 هي التي تكون جهة F

total deflection will be  $x_{eq}$

b. Parallel  $\rightarrow$



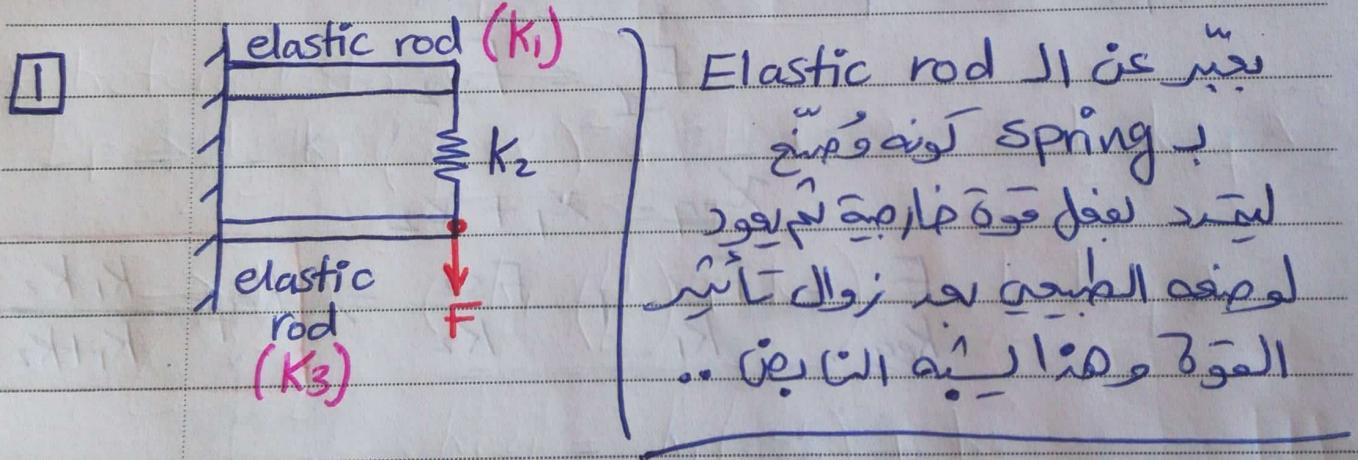
$$x_{eq} = x_1 = x_2$$

$$F = F_{s1} + F_{s2}$$

$$K_{eq} x_{eq} = k_1 x_1 + k_2 x_2$$

$$K_{eq} = k_1 + k_2$$

Ex : Find  $K_{eq}$  for the following :-

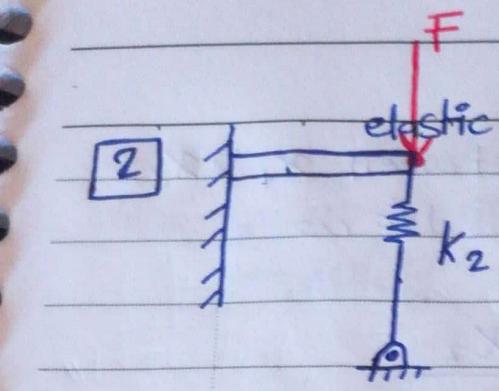


$$K_1 \text{ in series with } K_2 \rightarrow K_{12} = \frac{K_1 K_2}{K_1 + K_2}$$

$$K_{12} \text{ in parallel with } K_3 \rightarrow K_{eq} = K_{12} + K_3$$

$$K_{eq} = \frac{K_1 K_2}{K_1 + K_2} + K_3$$

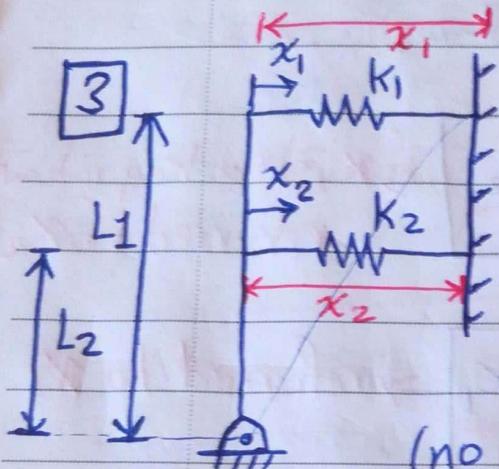
لذلك فعاليات  
:



No. \_\_\_\_\_

$k_1$  in parallel with  $k_2$

$K_{eq} = k_1 + k_2$



الفكرة: نعمل على إثبات مبرهنة  
كونية طاقة معنوية بالنظام  
عن طريق رسم خوارزمية  
الآن هو مثلك قانون الـ  
[Potential Energy]  
كونية عالي مركبة للناريين (no kinetic energy)

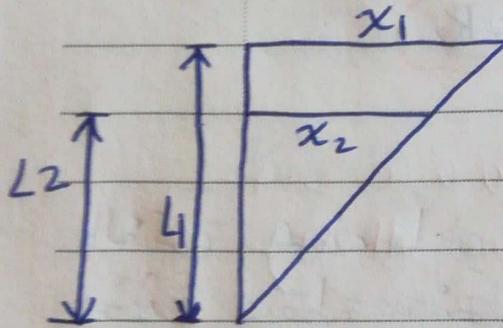
$$E_p = \frac{1}{2} k_1 (x_1)^2 + \frac{1}{2} k_2 (x_2)^2 = \frac{1}{2} K_{eq} x_{1,2}^2$$

علاقة: هنا النظائر عبارة عن 1DOF كونية المعنوية  
ستجدها في  $(x_2 \& x_1)$

DOF = No. of independent variables  
"motions"

بعضها البعض، يعني في دائرة سعى  
 $(x_2 \& x_1)$   
فيمكن وصف سلوك الأفراد ويعود صفات الدائرة إلى  $E_p$  لحساب  
بيانات الرؤى التي تسببت في ذلك  $x_2$  أو  $x_1$  ...

→ to find the relation between  $x_1$  &  $x_2$  :-  
 $\Delta L_1 \propto L_1$ ,  $\Delta L_2 \propto L_2$



$$\frac{x_1}{L_1} = \frac{x_2}{L_2}$$

$$\boxed{x_1 = \frac{L_1}{L_2} x_2} \rightarrow x_1 \text{ is } K_{eq} \text{ when referred to } x_2$$

or

$$\boxed{x_2 = \frac{L_2}{L_1} x_1} \rightarrow \text{referred to } x_1$$

\* Referred to  $x_2$  →

$$E_p \Rightarrow \frac{1}{2} K_1 \left( \frac{L_1}{L_2} x_2 \right)^2 + \frac{1}{2} K_2 x_2^2 = \frac{1}{2} K_{eq} x_2^2$$

$$\Rightarrow \cancel{\frac{1}{2} x_2^2} \left( K_1 \left( \frac{L_1}{L_2} \right)^2 + K_2 \right) = \cancel{\frac{1}{2} K_{eq} x_2^2}$$

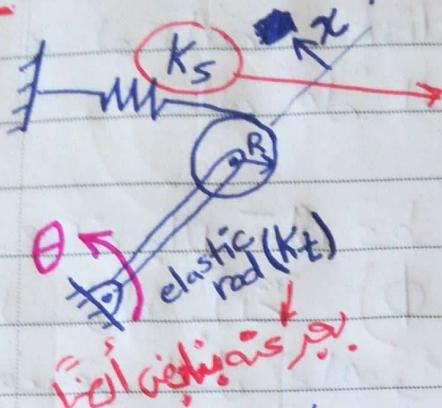
$$\therefore \boxed{K_{eq} = K_1 \left( \frac{L_1}{L_2} \right)^2 + K_2}$$

or

referred to  $x_1$  →

$$\boxed{K_{eq} = K_1 + \left( \frac{L_2}{L_1} \right)^2 K_2}$$

Ex: Find  $k_{eq}$  for the below system :-



(linearly) سرعة  
الربيع مع مرحلة  
disk الى تحلي لـ  
يضغط درجة حرارة

Here, there's 2 Variables  $\rightarrow x$   
 $\downarrow \theta$

1 DOF  $\leftarrow$  سرعة دوارة مع دوارة  
عندما تكون العلاقة بين  $\theta$  و  $x$  مترابطة  
كذلك مابين قوة مارجنة وعوارة

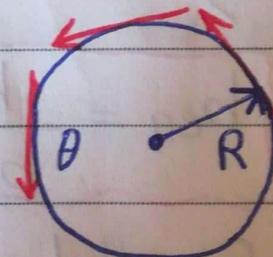
$$E_p = \frac{1}{2} k_t \theta^2 + \frac{1}{2} k_s x^2$$

\* For circular motion:

referred to  $\theta$ :

$$E_p \Rightarrow \frac{1}{2} k_t \theta^2 + \frac{1}{2} k_s (R^2 \theta^2)$$

$$= \frac{1}{2} k_{eq} \theta^2$$



$$V_{linear} = R\dot{\theta}$$

$$a = R\ddot{\theta}$$

$$\frac{1}{2} \theta^2 (k_t + R^2 k_s) = \frac{1}{2} k_{eq} \theta^2$$

$$so \Rightarrow \boxed{x = R\theta}$$

$$\therefore k_{eq} = k_t + R^2 k_s$$

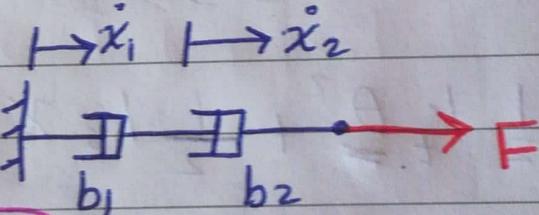
فم ← لو كانت الـ  $\phi$  تختلف عن disk فم ←  
يختلف حلّ السؤال

وحينما بالعادة  $\theta$  يساوي  $\phi$  لكن هنا من الممكن  
السؤال بأن  $\theta$   $\neq$   $\phi$  (  $\theta$  the same of the  $\phi$  )  
فيبقى الحل كما هو في الـ 15

# Damping element  $\rightarrow$  dissipates energy in  
the form of heat losses .

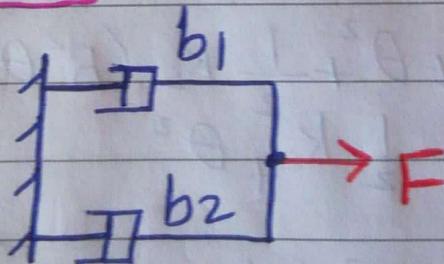
Combination ::

a.) Series



$$b_{eq} = \frac{b_1 b_2}{b_1 + b_2}$$

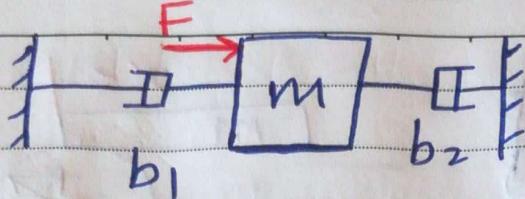
b.) Parallel



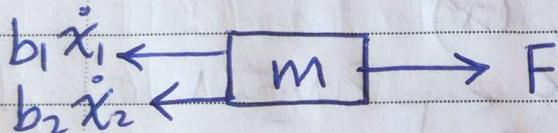
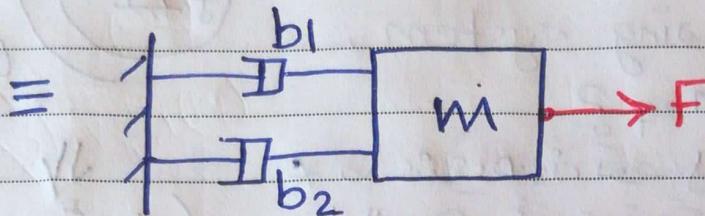
$$b_{eq} = b_1 + b_2$$

Ex

No.



## Parallel



الغرس توزيع على الـ 2 dampers

$$\therefore \text{beg} = b_1 + b_2$$

# Mass / Inertia  $\rightarrow$  resistance to acceleration,  
it stores Kinetic energy.

$$E_K = \frac{1}{2} m \dot{x}^2 = \frac{1}{2} J \dot{\theta}^2$$

بگو Keg و زیگر لامپر damper یا spring لامپر ایکیو

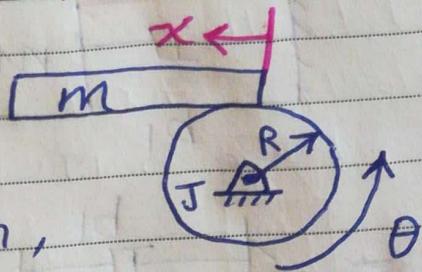
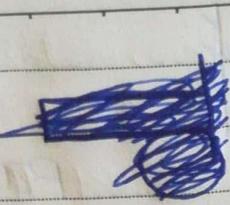
Series إذا كان موجيّاً (أي أن  $\omega$  موجيّة) لـ

أو  $E_p$ , Potential Energy, كنابي (عاليون) الـ parallel

$E_k$  + Kinetic Energy,  $\dot{m} \frac{1}{2} v^2$   $\rightarrow$   $J_{eq}$  مماثل  $\dot{m} \frac{1}{2} v^2$  (لكن في

يُبَلِّغُونَ بِكُلِّ مُعْنَىٰ وَمُفْعَلٍ بِالنَّهَايَةِ .. FIVE APPLE

Ex



For the following system,

find  $m_{eq}$ ?

إذا mass II just is in series to \*  
... Series of parallel

rotation law (disk)

يلقي مركبة الـ rod بكل خطٍ بغير

... (x)

بِسْمِ اللَّهِ الرَّحْمَنِ الرَّحِيمِ

$x$   $\theta$   $\text{Distanz}$

$$\begin{aligned} x &= R\theta \\ \dot{x} &= R\dot{\theta} \\ \ddot{x} &= R\ddot{\theta} \end{aligned}$$

∴ So the system is 1DOF

cuz there are 2 variables ~~but~~ dependent with each other ..

\* to find meg :-

$$E_K = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} J \dot{\theta}^2$$

referred to  $\ddot{x}$  :  $E_k = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} J \frac{\dot{x}^2}{R^2}$

$$= \frac{1}{2} M_{eq} \dot{x}^2$$

$$\frac{1}{2} \dot{x}^2 \left( m + \frac{J}{R^2} \right) = \frac{1}{2} m_{eq} \dot{x}^2$$

$$\therefore \boxed{m_{eq} = m + \frac{J}{R^2}}$$

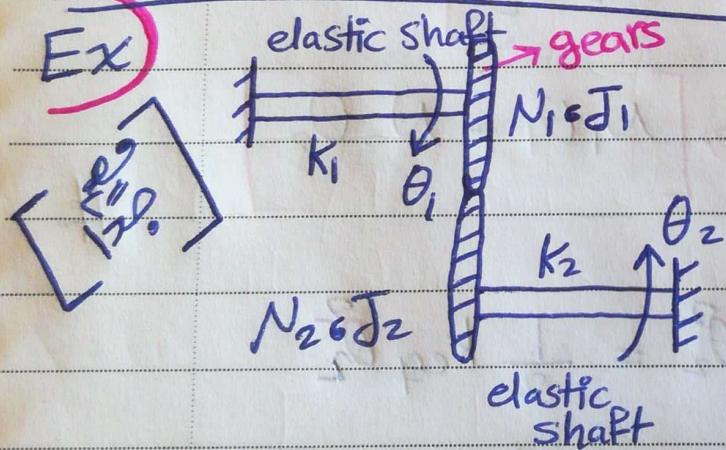
or

referred to  $\dot{\theta}$  :  $\frac{1}{2} m (R\dot{\theta})^2 + \frac{1}{2} J\dot{\theta}^2 = \frac{1}{2} m_{eq} \dot{\theta}^2$

$$\frac{1}{2} \dot{\theta}^2 (R^2 m + J) = \frac{1}{2} m_{eq} \dot{\theta}^2$$

$$\therefore \boxed{m_{eq} = R^2 m + J}$$

Ex



we have 2 gears

each have:  $N, J$ 

$[N: \text{no. of teeth}]$   
 $[J: \text{Inertia}]$

Find: ①  $k_{eq}$  ②  $J_{eq}$  ?

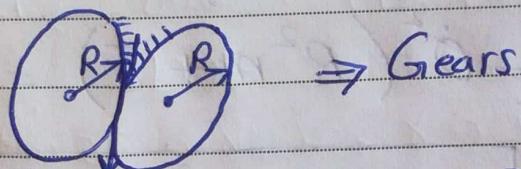
$\frac{1}{2} (J_{eq} \dot{\theta}^2)$  gear || go shaft || (العنان) || (العنان)  $\theta_1 \leftarrow$  في حين  $\theta_2 \leftarrow$   
 (أعواد)  $\theta_2$

$\frac{1}{2} (J_{eq} \dot{\theta}^2)$  gear || go shaft || (العنان) || (العنان)  $\theta_2 \leftarrow$   
 (أعواد)

وبالتالي نجد 2 gears || له 1 DOF ← (نظام)

Gear Ratio || يدل على  $\theta_2 \rightarrow \theta_1$  (نسبة)

لأعلى gear || يساوي لأسفل gear || \*  
مساوية مع السفلي gear ||



$$v(x) = R \dot{\theta} \quad \text{Motion طبقي Circular motion}$$

For Given  $R \rightarrow [R_1 \dot{\theta}_1 = R_2 \dot{\theta}_2]$

But, For Given  $N \rightarrow [N_1 \theta_1 = N_2 \theta_2]$   
as in this example

$$E_p = \frac{1}{2} k_1 \dot{\theta}_1^2 + \frac{1}{2} k_2 \dot{\theta}_2^2 = \frac{1}{2} k_{eq} \dot{\theta}_2^2$$

~~$$\frac{1}{2} k \left( \frac{N_2}{N_1} \right)^2 \dot{\theta}_2^2 + \frac{1}{2} k_2 \dot{\theta}_2^2$$~~

$$\cancel{\frac{1}{2} \dot{\theta}_2^2} \left( k_1 \left( \frac{N_2}{N_1} \right)^2 + k_2 \right) = \cancel{\frac{1}{2} k_{eq} \dot{\theta}_2^2}$$

$$\therefore \left[ k_{eq} = k_1 \left( \frac{N_2}{N_1} \right)^2 + k_2 \right]$$

referred to  $\theta_2$

$$\underline{\underline{2}} \quad E_k = \frac{1}{2} J_1 \dot{\theta}_1^2 + \frac{1}{2} J_2 \dot{\theta}_2^2 = \frac{1}{2} J_{eq} \dot{\theta}_2^2$$

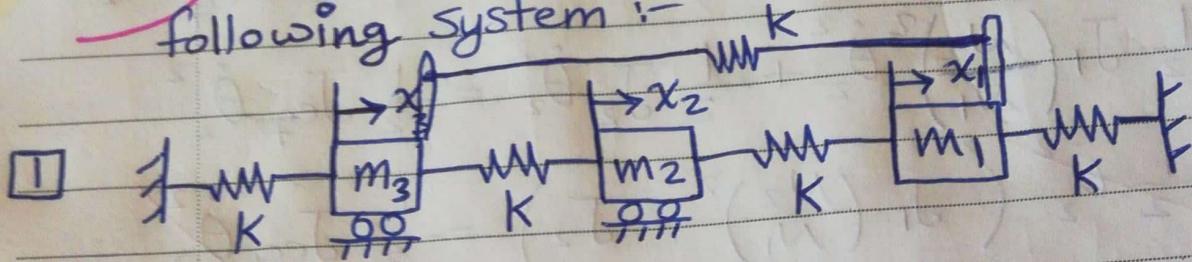
$$\frac{1}{2} J_1 \left( \frac{N_2}{N_1} \right)^2 \dot{\theta}_2^2 + \frac{1}{2} J_2 \dot{\theta}_2^2 = \frac{1}{2} J_{eq} \dot{\theta}_2^2$$

$$\frac{1}{2} \cancel{\dot{\theta}_2^2} \left( J_1 \left( \frac{N_2}{N_1} \right)^2 + J_2 \right) = \frac{1}{2} J_{eq} \cancel{\dot{\theta}_2^2}$$

$$\therefore \boxed{J_{eq} = J_1 \left( \frac{N_2}{N_1} \right)^2 + J_2}$$

Degree of Freedom (DOF)  $\equiv$  The minimum number of independent coordinates that can specify the position of the system completely...

Ex) Derive the equation of motion for the following system :-



$x_3, x_2, x_1$   $\leftarrow$  3 DOF

new 3 axes are taken

Lagrange  $\leftarrow$  1st law of motion  
FBD  $\leftarrow$

FBD

$$m_3 \ddot{x}_3 + kx_3 + k(x_3 - x_2) + k(x_3 - x_1) = 0$$

$$m_2 \ddot{x}_2 + k(x_2 - x_3) + k(x_2 - x_1) = 0$$

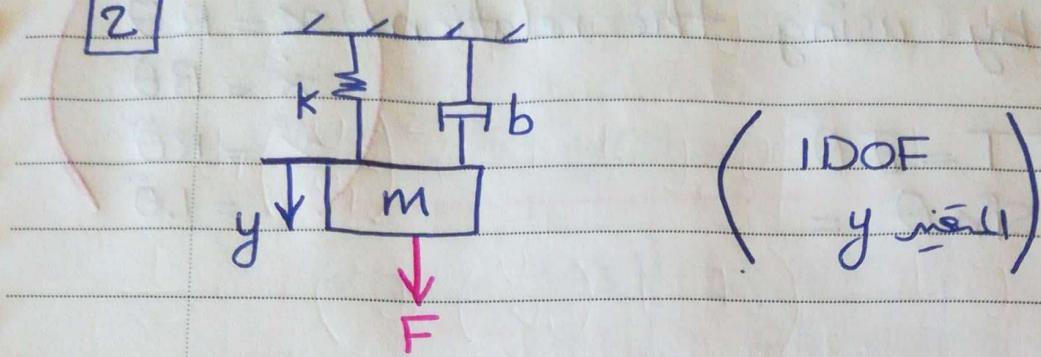
$$m_1 \ddot{x}_1 + k(x_1 - x_3) + k(x_1 - x_2) + kx_1 = 0$$

new axes are taken  $\rightarrow$  3 DOF

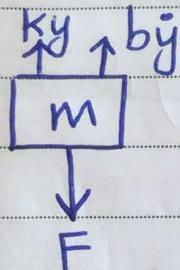
for given  $m, k$

to find  $x_1/x_2/x_3$

2



FBD :



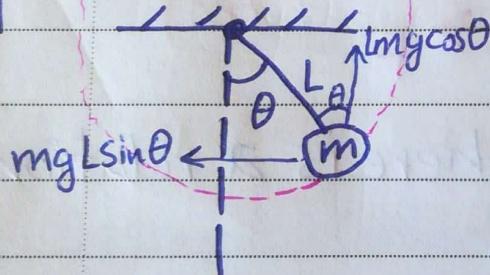
$$\sum F_x = \emptyset$$

$$\sum F_y = m\ddot{y} \rightarrow$$

$$m\ddot{y} + Ky + by = F$$

Static Deformation | مطلب تقویت العزم لبیی \*

3



$$\text{If } J = mL^2 \text{ :}$$

$$\sum T = J\ddot{\theta}$$

$$-mgL\sin\theta = mL^2\ddot{\theta}$$

$$\frac{-g\sin\theta}{L} = \frac{J\ddot{\theta}}{K}$$

$$-\frac{g}{L}\sin\theta = \ddot{\theta} \rightarrow \text{non-linear system}$$

After Linearization :

$$-\frac{g}{L}\widehat{\theta} = \ddot{\theta}$$

$$\sin\theta \approx \theta$$

$$\ddot{\theta} + \frac{g}{L}\theta = \emptyset$$

Or by using the relation

$$x = R\theta$$

$$\dot{x} = R\dot{\theta}$$

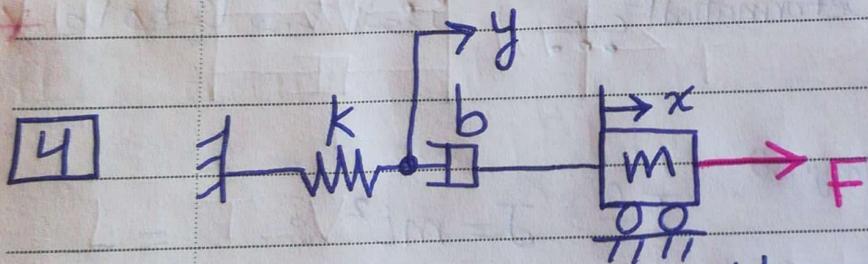
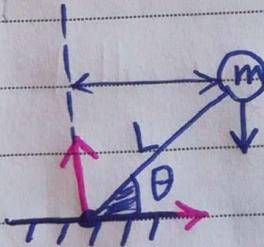
$$\ddot{x} = R\ddot{\theta}$$

$$= L\ddot{\theta}$$

$$\sum T = J\ddot{\theta}$$

$$-mgL \sin \theta =$$

# Inverted Pendulum  $\rightarrow$



1DOF system because there is a relationship between y and x ...

$$m\ddot{x} + b(\dot{x} - \dot{y}) = F \quad \text{--- (1)}$$

$$-ky - b(\dot{y} - \dot{x}) = 0$$

$$b(\dot{y} - \dot{x}) = -ky$$

$$b(\dot{x} - \dot{y}) = ky \quad \boxed{\text{substitute in (1)}}$$

$$m\ddot{x} + ky = F$$

by taking the laplace for ②, we get:

$$bsx(s) - bs y(s) = k y(s)$$

$$x(s) = y(s) \left[ \frac{bs+k}{bs} \right]$$

مُعَادِلَةِ الْجَهَةِ الْأَنْتَرِيَوْرِيَّةِ مُعَادِلَةِ الْجَهَةِ الْأَنْتَرِيَوْرِيَّةِ  
مُعَادِلَةِ الْجَهَةِ الْأَنْتَرِيَوْرِيَّةِ

# Deriving the differential equation  
using lagrange method (FBD)  $\rightarrow$   $L = T - V$

Lagrange Method :  $L = T - V$

$T \equiv$  kinetic energy (translation + Rotation)

$V \equiv$  Potential energy (Spring + gravity)

مُسْكِنُ كِبِيرٍ بِإِنْسَانٍ  $\rightarrow$  مُسْكِنُ كِبِيرٍ بِإِنْسَانٍ

DE  $\rightarrow$  مُعَادِلَةِ الْجَهَةِ الْأَنْتَرِيَوْرِيَّةِ (L = T - V) مُعَادِلَةِ الْجَهَةِ الْأَنْتَرِيَوْرِيَّةِ

$$\frac{\partial}{\partial t} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = F_{nc}$$

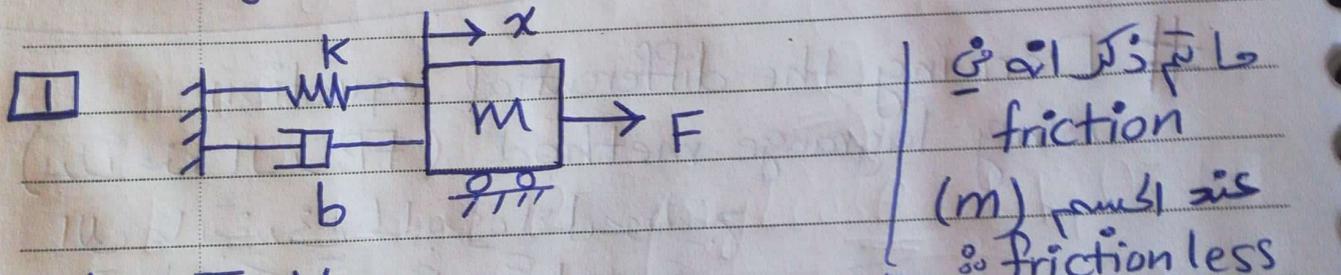
مُسْكِنُ كِبِيرٍ بِإِنْسَانٍ  $\rightarrow$  مُسْكِنُ كِبِيرٍ بِإِنْسَانٍ

where  $F_{nc}$  = Non conservative force  
(damping, friction, external force)

لهم  $\ddot{x}$  هو المموجة المموجة

$q_i$  = system variable

**Ex** Derive the DE for the following systems  
using Lagrange method :-



$$L = T - V$$

$$T = \frac{1}{2} m v^2 = \frac{1}{2} m \dot{x}^2 \rightarrow \text{mass } \mathcal{H}$$

$$V = \frac{1}{2} K x^2 \rightarrow \text{موجة}$$

$$\therefore L = \frac{1}{2} m \dot{x}^2 - \frac{1}{2} K x^2$$

$$\frac{\partial L}{\partial \dot{x}} = m \dot{x} \rightarrow \text{أي رسمة موجة} \dot{x} \text{ هي}$$

$$\frac{\partial (m \dot{x})}{\partial t} = m \ddot{x} \rightarrow \text{we talk about } m \dot{x}(t) \xrightarrow{\text{موجة}} m \ddot{x}(t) \xrightarrow{\text{موجة}}$$

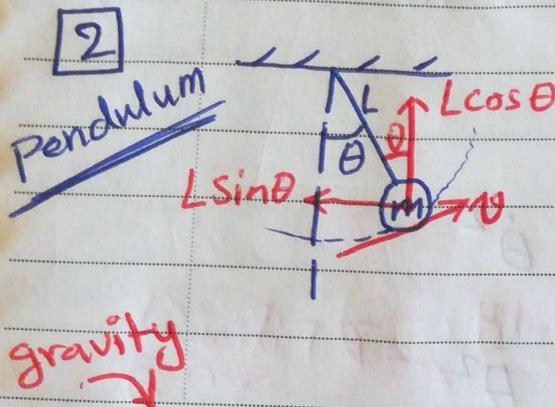
$$\frac{\partial L}{\partial x} = -Kx$$

$$m\ddot{x} + kx = F - b\dot{x}$$

External Force

damper

$$\therefore m\ddot{x} + kx + b\dot{x} = F$$



1 Variable ( $\theta$ ) / 1 DOF

$$L = T - V$$

$$T = \frac{1}{2} m v^2$$

$$= \frac{1}{2} m (L\dot{\theta})^2$$

$v = \dot{x}$

$\dot{x} = L\dot{\theta}$

$$V = -mgL \cos \theta$$

$$L = \frac{1}{2} mL^2 \dot{\theta}^2 + mgL \cos \theta$$

$$\frac{\partial L}{\partial \dot{\theta}} = mL^2 \dot{\theta}$$

$$\frac{\partial (mL^2 \dot{\theta})}{\partial t} = mL^2 \ddot{\theta}$$

$\ddot{\theta}$

$$\frac{\partial L}{\partial \theta} = -mgL \sin \theta$$

$$\approx -mgL\theta$$

Taylor Series

for small motion

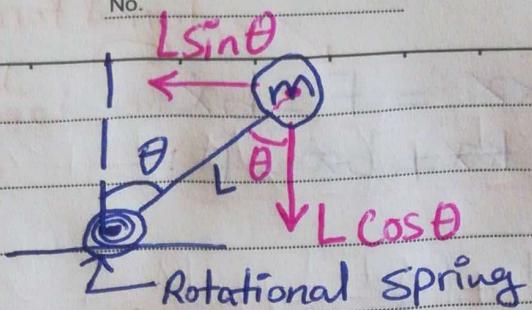
$\sin \theta \approx \theta$

$$\therefore mL^2 \ddot{\theta} + mgL\theta = \phi$$

$$L\ddot{\theta} + g\theta = \phi \rightarrow \ddot{\theta} + \frac{g}{L}\theta = \frac{\phi}{L}$$

## Inverted Pendulum

3



$$\vartheta = L\dot{\theta}$$

1 variable ( $\beta$ )/1 DOF

$$L = T - V$$

$$T = \frac{1}{2} m v^2 = \frac{1}{2} m L^2 \dot{\theta}^2$$

$$V = \frac{1}{2}mgL\cos\theta + \frac{1}{2}k_T\theta^2$$

gravity

Rotational spring

above  
the Ref.

$$L = \frac{1}{2} m L^2 \dot{\theta}^2 - mgL \cos\theta = -\frac{1}{2} k_t \theta^2$$

$$\frac{\partial L}{\partial \dot{\theta}} = mL^2 \ddot{\theta}$$

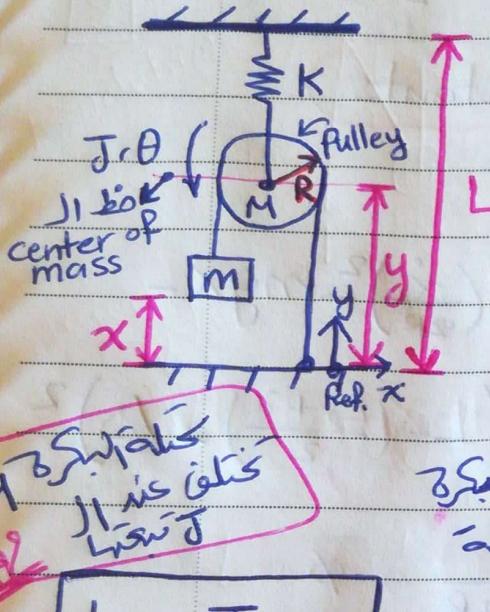
$$\frac{\partial (mL^2\ddot{\theta})}{\partial t} = mL^2\ddot{\theta}$$

$$\frac{\partial L}{\partial \theta} = mgL \sin \theta - \frac{1}{2} K_t \theta = mgL \theta - \frac{1}{2} K_t \theta$$

$$mL^2 \ddot{\theta} - mgL\theta + \frac{1}{2} k_t \theta = \phi$$

Ex Derive the DE of the following system using lagrange method :-

1 Spring Free Length ( $\Delta$ )



we have here 3 variables  
but all in 1 DOF  
 $x, y, \theta$

$$y = R\theta \rightarrow \dot{y} = R\dot{\theta} \quad \text{--- (1)}$$

$$L_0 = y + (y - x) = 2y - x \quad \text{--- (2)}$$

$$L = T - V$$

$$T = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} M \dot{y}^2 + \frac{1}{2} J \dot{\theta}^2$$

For gravity

$$V = mg \cancel{x} + Mgy + \frac{1}{2} k ((L_0 - y) - \Delta)^2$$

cancel Ref. is null

الآن كل وظائف  
ستغيرها بعدين  
نحو مجموع  
ذراء  
الثانية

where  $\dot{y}$  : translational speed

$\dot{\theta}$  : Rotational speed

$\dot{x}$  gives Lagrange equ. II due to  $\dot{x}$  and  $\dot{y}$   $\rightarrow$  (1) gives  $\dot{\theta}$

$$\text{From (2)} \rightarrow y = \frac{L_0 + x}{2}, \dot{y} = \frac{\dot{x}}{2}$$

(1) gives

$$\frac{\dot{x}}{2} = R\dot{\theta} \rightarrow \dot{\theta} = \frac{\dot{x}}{2R}$$

$$T = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} M \left(\frac{\dot{x}}{2}\right)^2 + \frac{1}{2} J \left(\frac{\ddot{x}}{2R}\right)^2$$

$$V = mgx + Mg\left(\frac{L_0+x}{2}\right) + \frac{1}{2}k\left(L - \left(\frac{L_0+x}{2}\right) - \Delta\right)^2$$

$L, L_0, \Delta \Rightarrow$  Constants

$$\Rightarrow L = T - V$$

$$= \frac{1}{2} m \dot{x}^2 + \frac{1}{2} M \left(\frac{\dot{x}}{2}\right)^2 + \frac{1}{2} J \left(\frac{\ddot{x}}{2R}\right)^2 - mgx -$$

$$Mg\left(\frac{L_0+x}{2}\right) - \frac{1}{2} k\left(L - \left(\frac{L_0+x}{2}\right) - \Delta\right)^2$$

$$\frac{\partial L}{\partial \dot{x}} = m\dot{x} + M\frac{\dot{x}}{2} + J\frac{\ddot{x}}{2R \times 2R} - \phi$$

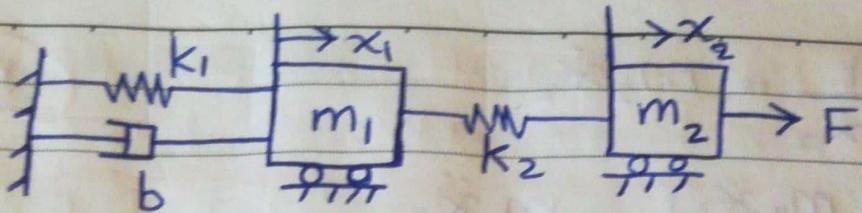
$$\frac{\partial L}{\partial t} \left( \frac{\partial L}{\partial \dot{x}} \right) = m\ddot{x} + \frac{M}{4}\ddot{x} + J\frac{\ddot{x}}{4R^2}$$

$$\frac{\partial L}{\partial x} = -(mg) - \left(Mg \times \frac{1}{2}\right) + \frac{1}{2}k\left(L - \frac{L_0+x}{2} - \Delta\right)$$

$$\therefore \phi = m\ddot{x} + \frac{M}{4}\ddot{x} + J\frac{\ddot{x}}{4R^2} + mg + \frac{M}{2}g - \frac{1}{2}k\left(L - \frac{L_0+x}{2} - \Delta\right)$$

$\phi = \ddot{x}\left(m + \frac{M}{4} + \frac{J}{4R^2}\right) + g\left(m + \frac{M}{2}\right) - \frac{1}{2}k\left(L - \frac{L_0+x}{2} - \Delta\right)$

[2]



In this system, there's 2 variables / 2DOF cuz they're independent variables ( $x_1$  &  $x_2$ )

$x_2, x_1$  جانبیان Lagrange II ترکیب معاویت می‌شوند

$$T = \frac{1}{2} m_1 \dot{x}_1^2 + \frac{1}{2} m_2 \dot{x}_2^2$$

$$V = \frac{1}{2} k_1 x_1^2 + \frac{1}{2} k_2 (x_1 - x_2)^2$$

$$\therefore L = T - V$$

$$= \frac{1}{2} m_1 \dot{x}_1^2 + \frac{1}{2} m_2 \dot{x}_2^2 - \frac{1}{2} k_1 x_1^2 - \frac{1}{2} k_2 (x_1 - x_2)^2$$

$$\frac{\partial}{\partial t} \left( \frac{\partial L}{\partial \dot{x}_1} \right) - \frac{\partial L}{\partial x_1} = -b \dot{x}_1$$

$m_1 \ddot{x}_1$

$$\frac{\partial L}{\partial x_1} = -k_1 x_1 - k_2 (x_1 - x_2)$$

$$\frac{\partial L}{\partial \dot{x}_1} = m_1 \dot{x}_1 \rightarrow \frac{\partial (m_1 \dot{x}_1)}{\partial t} = m_1 \ddot{x}_1$$

$$m_1 \ddot{x}_1 - (-k_1 x_1 - k_2 (x_1 - x_2)) = -b \dot{x}_1$$

$$(m_1 \ddot{x}_1 + k_1 x_1 + k_2 (x_1 - x_2) + b \dot{x}_1) = 0$$

$$\frac{\partial L}{\partial \dot{x}_2} = m_2 \ddot{x}_2$$

$m_2 \ddot{x}_2$

$$\frac{\partial (m_2 \dot{x}_2)}{\partial t} = m_2 \ddot{x}_2$$

$$\frac{\partial L}{\partial x_2} = +k_2 (x_1 - x_2)$$

$$m_2 \ddot{x}_2 - k_2 (x_1 - x_2) = F$$

⇒ If we say that :

$$L = \frac{1}{2} m_1 \dot{x}_1^2 + \frac{1}{2} m_2 \dot{x}_2^2 - \frac{1}{2} k_1 x_1^2 - \frac{1}{2} k_2 (x_2 - x_1)^2$$

$$\text{For } m_1 \text{ so } m_1 \ddot{x}_1 = \frac{\partial}{\partial t} (m_1 \dot{x}_1)$$

$$\frac{\partial L}{\partial x_1} = -k_1 x_1 + k_2 (x_2 - x_1)$$

$$m_1 \ddot{x}_1 + k_1 x_1 - k_2 (x_2 - x_1) = -b \dot{x}_1$$

the same as before #

$$m_1 \ddot{x}_1 + b \dot{x}_1 + k_1 x_1 + k_2 (x_1 - x_2) = \phi$$

$$\text{For } m_2 \text{ so } \frac{\partial}{\partial t} (m_2 \dot{x}_2) = m_2 \ddot{x}_2$$

$$\frac{\partial L}{\partial x_2} = -k_2 (x_2 - x_1)$$

$$m_2 \ddot{x}_2 + k_2 (x_2 - x_1) = F$$

also, the same as before #

FIVE APPLE

## Ch. 4 & Electrical System

- Basic Elements
  - Resistor
  - Capacitor
  - Inductor
- Describing Variables
  - charge
  - current
  - voltage

II Capacitor :-  $\frac{1}{1} C$

مختبر  
الذرة  
التيار  
الطاقة

$$U(t) = \frac{1}{C} \int I \cdot dt = \frac{Q}{C} \rightarrow \text{charge}$$

$I = \frac{dQ}{dt}$

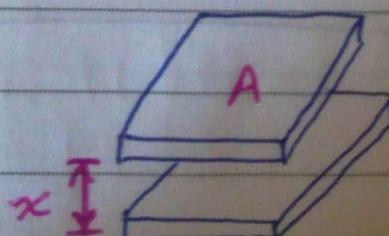
$$E_c = \frac{1}{2} C V^2 = \frac{1}{2} \frac{Q^2}{C}$$

$$I(t) = C \frac{dU_c}{dt}$$

ex 1 | Capacitor for two parallel plates

$$C = \frac{\epsilon A}{x}$$

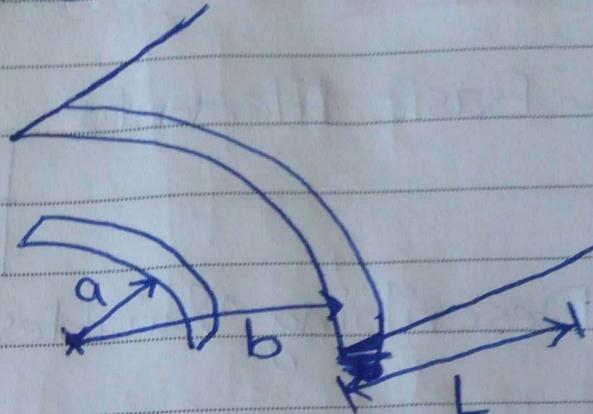
$\epsilon$  & dielectric  
constant



ex2 | Capacitor for two concentric cylindrical

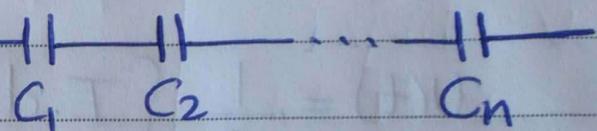
$$C = \frac{2\pi\epsilon_0 L}{\ln\left(\frac{b}{a}\right)}$$

where ( $b \gg a$ )



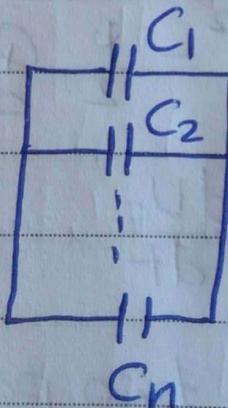
# Combinations  $\infty$

a. Series  $\rightarrow$

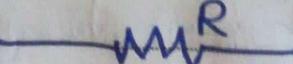


$$\frac{1}{C_{eq}} = \sum_{i=1}^n \frac{1}{C_i}$$

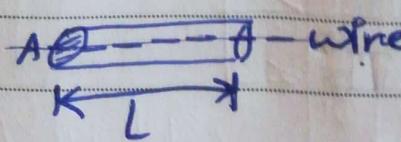
b. Parallel  $\rightarrow$



$$C_{eq} = \sum_{i=1}^n C_i$$

2 Resistor :- 

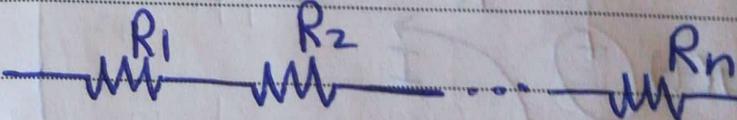
$$V(t) = IR$$

~~A~~   $R = \rho L / A$

$\rho$ : electric resistivity

$$\text{Power dissipated} = IV = \frac{V^2}{R} = I^2 R$$

# Combinations  $\circlearrowright$

a. Series  $\rightarrow$  

$$R_{\text{eq}} = \sum_{i=1}^n R_i$$

b. Parallel



$$\frac{1}{R_{\text{eq}}} = \sum_{i=1}^n \frac{1}{R_i}$$

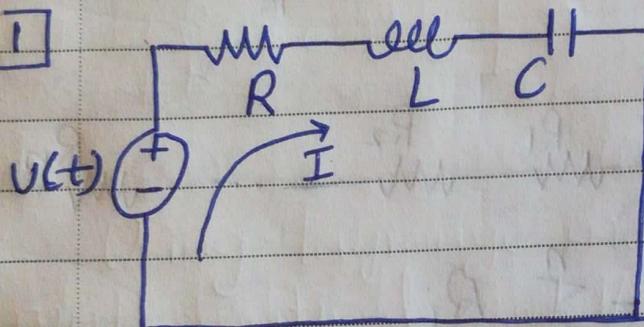
3 Inductor :-

$$V(t) = L \frac{dI}{dt} = L \ddot{Q}(t)$$

$$E_L = \frac{1}{2} L I^2$$

Ex

1



Output : I

Input : V

$$I = I_R = I_C = I_L$$

a.  $V(t) = V_R + V_L + V_C$

$$V = IR + L \frac{dI}{dt} + \frac{1}{C} \int I \, dt$$

Differential Equ.  $\leftarrow$  (i, S-1)

ci  $\ddot{V}$   $\ddot{I}$   $\dot{I}$   $I$

$$\ddot{V} = IR + L \ddot{I} + \frac{1}{C} I^*$$

$$S \ddot{V}(S) = S I(S) R + S^2 L I(S) + \frac{1}{C} I(S)$$

$$S \ddot{V}(S) = I(S) \left[ R S + S^2 L + \frac{1}{C} \right]$$

$$\frac{I(S)}{V(S)} = \frac{S}{R S + L S^2 + \frac{1}{C}}$$

b. using lagrange method  $\rightarrow$

$$\boxed{L = T - V} \rightarrow T \text{ of inductors}$$

$V$  of Capacitors

non-conservative  $\text{g}(V/I)$   
force sources

+  
Resistor  
Voltage  
 $IR$

$$L = \frac{1}{2} L \dot{Q}^2 - \frac{1}{2C} Q^2$$

$$\frac{\partial}{\partial t} \left( \frac{\partial L}{\partial \dot{Q}} \right) - \frac{\partial L}{\partial Q} = F_{nc} \Rightarrow$$

$$\frac{\partial L}{\partial \dot{Q}} = L \dot{Q}$$

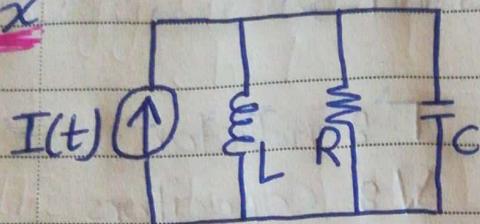
$$\frac{\partial (L \dot{Q})}{\partial t} = L \ddot{Q}$$

$$\frac{\partial L}{\partial Q} = -\frac{1}{C} Q$$

$$\therefore L \ddot{Q} - \left( -\frac{Q}{C} \right) = V(t) - R \dot{Q}$$

$$\boxed{L \ddot{Q} + R \dot{Q} + \frac{1}{C} Q = V(t)} \rightarrow RI$$

حالات - انماليات كالمواء المعدية في الاتصال  
وهي احاد المصال

Ex

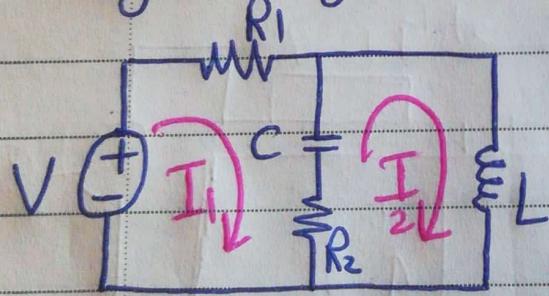
$I$  : input  
 $V$  : output

$$I(t) = I_R + I_C + I_L$$

$$= \frac{V}{R} + C \frac{dV_C}{dt} + \frac{1}{L} \int V(t) \cdot dt$$

$$\dot{I} = \frac{\dot{V}}{R} + C \ddot{V} + \frac{1}{L} V$$

Ex Using lagrange method, derive the mathematical modelling for the following system?



$$I_1 \equiv Q_1$$

$$I_2 \equiv Q_2$$

Input ( $V$ )

Output ( $Q_1 / Q_2$ )

"indep. variables,  $Q_1$  &  $Q_2$ "

$$L = T - V$$

$$= \frac{1}{2} L \dot{Q}_2^2 - \frac{1}{2C} (Q_1 - Q_2)^2$$

$$\rightarrow \frac{\partial(\frac{\partial L}{\partial \dot{Q}_1})}{\partial t} - \frac{\partial L}{\partial Q_1} = \vartheta(t) - R_1 \dot{Q}_1 - R_2 (\dot{Q}_1 - \dot{Q}_2)$$

$$\varphi - \left( -\frac{1}{C} (Q_1 - Q_2) \right) = \vartheta(t) - R_1 \dot{Q}_1 - R_2 (\dot{Q}_1 - \dot{Q}_2)$$

$$\left[ \frac{Q_1 - Q_2}{C} = \vartheta(t) - R_1 \dot{Q}_1 - R_2 (\dot{Q}_1 - \dot{Q}_2) \right]$$

$$\rightarrow \frac{\partial(\frac{\partial L}{\partial \dot{Q}_2})}{\partial t} - \frac{\partial L}{\partial Q_2} = -R_2 (\dot{Q}_2 - \dot{Q}_1)$$

$$L \ddot{Q}_2 - \left( \frac{1}{C} (Q_1 - Q_2) \right) = -R_2 (\dot{Q}_2 - \dot{Q}_1)$$

$$\left[ L \ddot{Q}_2 - \frac{1}{C} (Q_1 - Q_2) = -R_2 (\dot{Q}_2 - \dot{Q}_1) \right]$$

## Ch.10 Analogy between mechanical and electrical system

1 Force - voltage analogy

$$\text{Force} = V$$

$$m = L$$

$$b = R$$

$$k = \frac{1}{C}$$

2 Force - current analogy

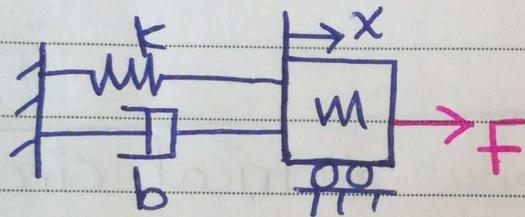
$$\text{Force} = I$$

$$m = C$$

$$b = \frac{1}{R}$$

$$k = \frac{1}{L}$$

Ex)

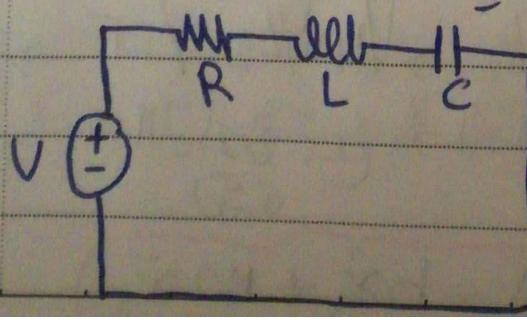


a.) Using Force voltage analogy?

$$\frac{m\ddot{x}}{L} + \frac{b\dot{x}}{R} + \frac{kx}{C} = \frac{f(t)}{V}$$

بعد تطبيق المبدأ  
الميكانيكي لقانون آلة  
محاكاة في النظام  
الكهربائي المعاكس

\* القطبية توزع وهذا يكون بموجب مبدأ المعاكس  
\* الكهربائية في المعاكس \*

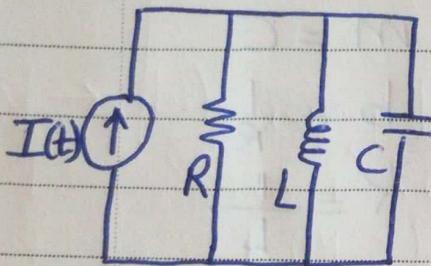


b.) Using force-current analogy?

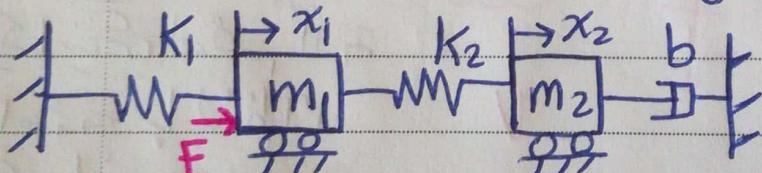
$$\underline{m} \ddot{x} + \underline{b} \dot{x} + \underline{k} x = \underline{F}$$

$$C \quad \frac{1}{R} \quad \frac{1}{L} \quad \frac{1}{I}$$

\* الآن يمكن تطبيق المبدأ



Ex Draw the analogous electrical circuit using : a- Force-current  
b- Force-voltage



$$m_1 \ddot{x}_1 + k_1 x_1 + k_2 (x_1 - x_2) = F \quad \text{--- (1)}$$

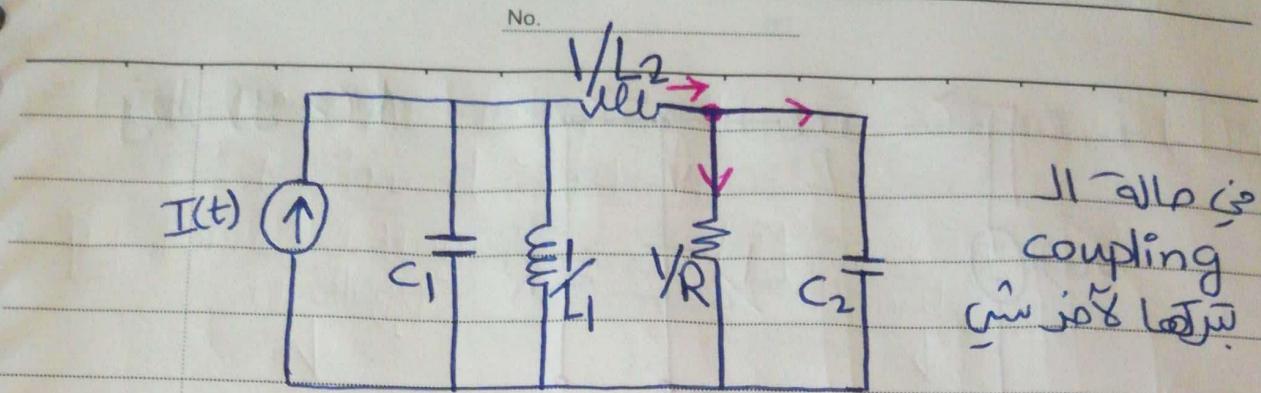
$$m_2 \ddot{x}_2 + b \dot{x}_2 + k_2 (x_2 - x_1) = \phi \quad \text{--- (2)}$$

a)  $m_1, m_2 \rightarrow C_1, C_2$  /  $k_1, k_2 \rightarrow \frac{1}{R_1}, \frac{1}{L_2}$  /  $F \rightarrow I(t)$

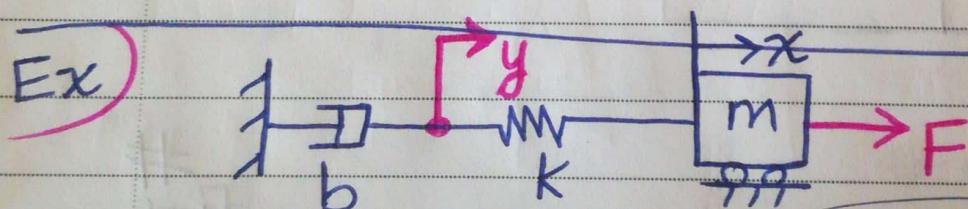
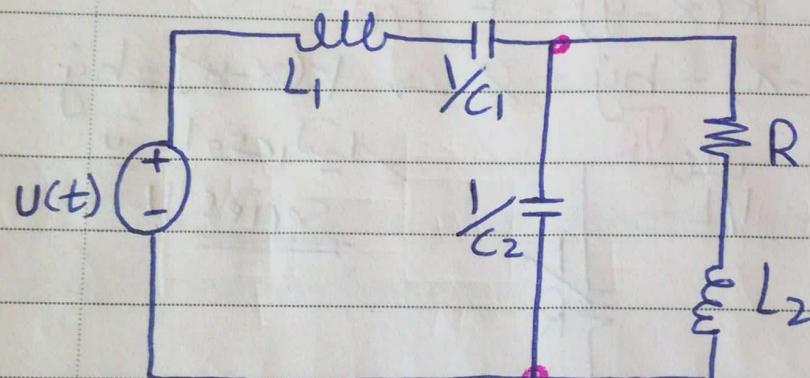
From (1)  $\Rightarrow$  الآن يمكن تطبيق المبدأ

( $\Rightarrow$  الآن يمكن تطبيق المبدأ) From (2) الآن يمكن تطبيق المبدأ  
coupling

$$k_2 (x_1 - x_2) = b \dot{x}_2 + m_2 \ddot{x}_2$$



b)  $m_1, m_2 \rightarrow L_1, L_2 / k_1, k_2 \rightarrow \frac{1}{C_1}, \frac{1}{C_2}$   
 $b \Rightarrow R / F \rightarrow V(t)$



$$F = m\ddot{x} + k(x - y) \quad \text{①}$$

$$-k(y - x) - b\dot{y} = \phi \quad \text{②}$$

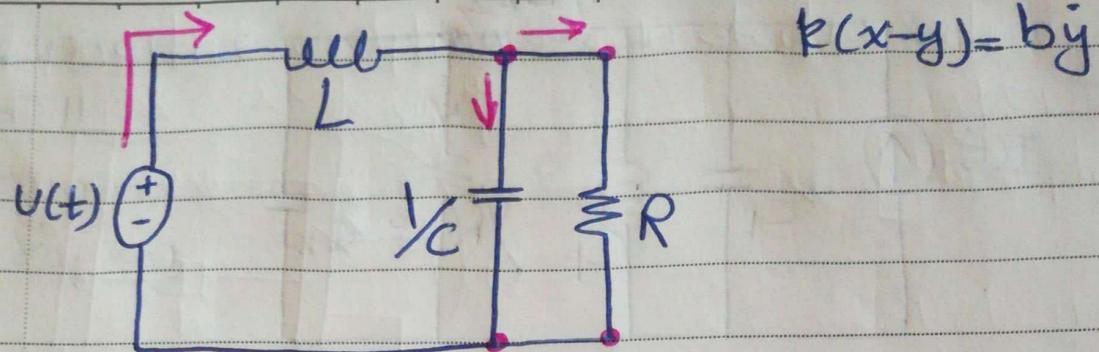
$k\Delta x \leftarrow [m] \rightarrow F$

$k(y - x)$   
 $b\dot{y}$

FBD

a.) Using force-voltage so

$$F \rightarrow V / m \rightarrow L / k \rightarrow \frac{1}{C} / b \rightarrow R$$

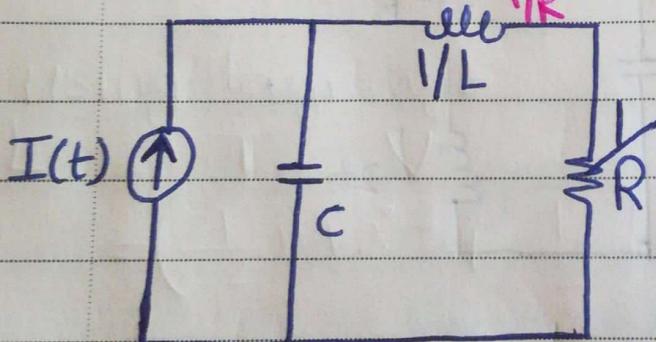


$$k(x-y) = bij$$

Ex → b.) using force current  $\mathbf{g}$

$$c m \ddot{x} + k \frac{V_L}{R} (x-y) = F I(t)$$

$$-k(y-x) - bij = 0 \rightarrow k(x-y) = bij$$

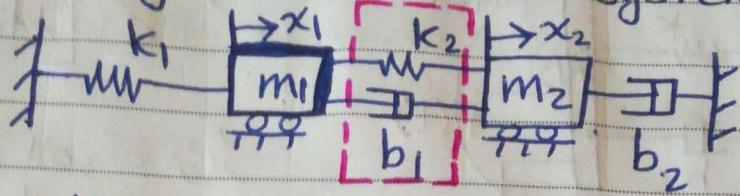


Series  $\frac{V_L}{R}$

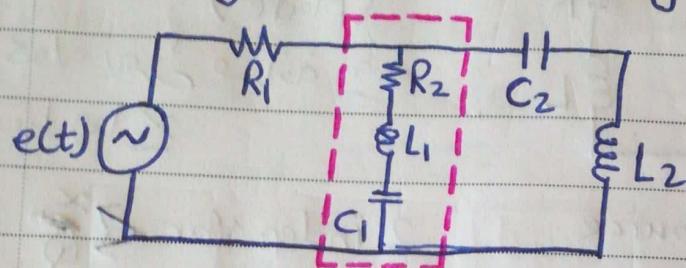
#  
First  
Exam

# Coupling and combined systems →

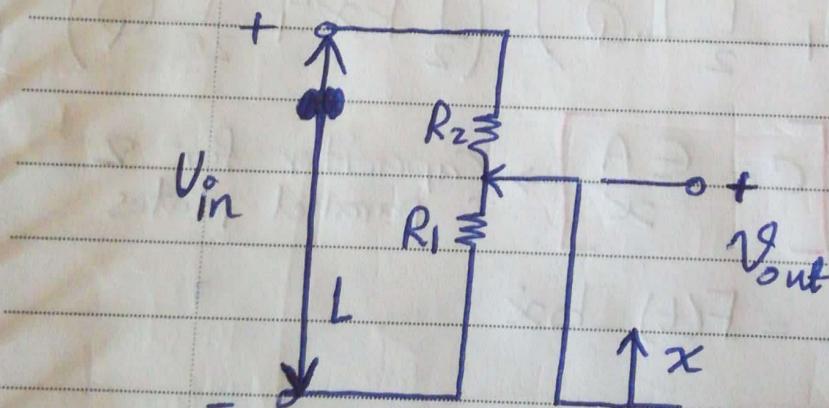
1] Coupling in mechanical system



2] Coupling in electrical system

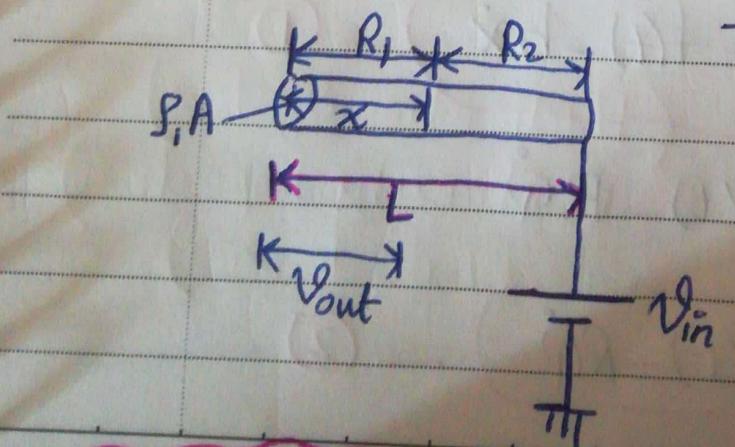


3] Resistive coupling electromechanical system  
e.g. potentiometer



$$* V_{out} = \frac{R_1}{R_1 + R_2} V_{in}$$

~~Resistive coupling~~



$$* V_{out} = \frac{x}{L} V_{in}$$

$$R_1 = \frac{\rho x}{A}$$

$R = \frac{\rho L}{A}$

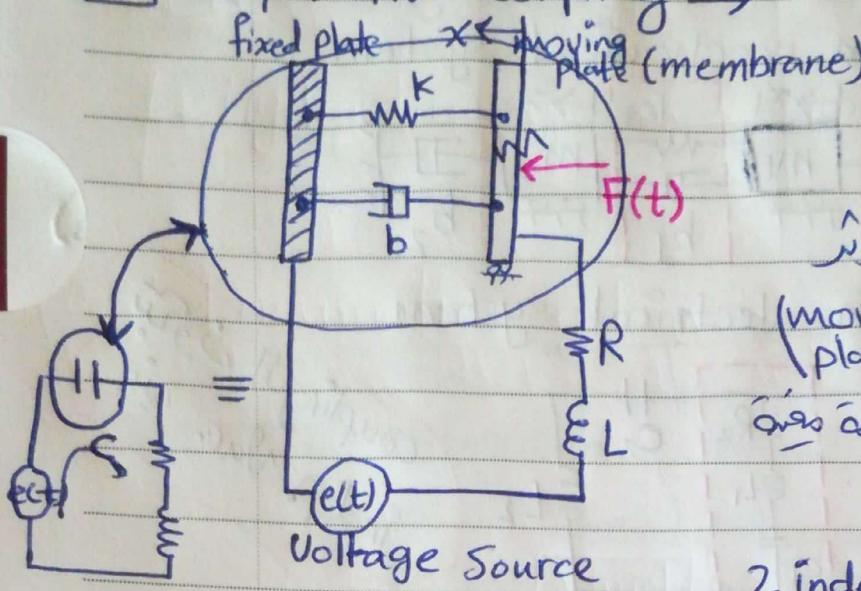
$R_2 = \frac{\rho (L-x)}{A}$  FIVE APPLE

# # Electromechanical Sys.

Resistive  
Capacitive  
Inductive

No.

4 Capacitive Coupling  $\rightarrow$  in electromechanical sys.



$\frac{dE}{dx}$   
[Microphone]

المحاذ العلوي  
(moving) لـ (de  
plate)  $\hat{x}$   
أو صافحة متحركة  
 $\hat{x}$  لـ  $\hat{x}$

2 indep. Var  $\begin{pmatrix} x \\ Q \end{pmatrix}$

using lagrange:

$$L = T - V$$

$$= \left( \frac{1}{2} m \dot{x}^2 + \frac{1}{2} L \dot{Q}^2 \right) - \left( \frac{1}{2} k x^2 + \frac{1}{2C} Q^2 \right)$$

$T$  : mass / Inductor  
 $V$  : Spring / Capacitor

where  $C = \frac{\epsilon A}{x}$   $\rightarrow$  Capacitor for 2 parallel plates

$$* \frac{\partial}{\partial t} \left( \frac{\partial L}{\partial \dot{x}} \right) - \frac{\partial L}{\partial x} = F(t) - b \dot{x}$$

$$m \ddot{x} + kx + \frac{Q^2}{2\epsilon A} = f(t) - b \dot{x} \quad \text{--- (1)}$$

$$* \frac{\partial}{\partial t} \left( \frac{\partial L}{\partial \dot{Q}} \right) - \frac{\partial L}{\partial Q} = e(t) - R \dot{Q}$$

$$L \ddot{Q} + \left( \frac{x}{\epsilon A} \right) Q = e(t) - R \dot{Q} \quad \text{--- (2)}$$

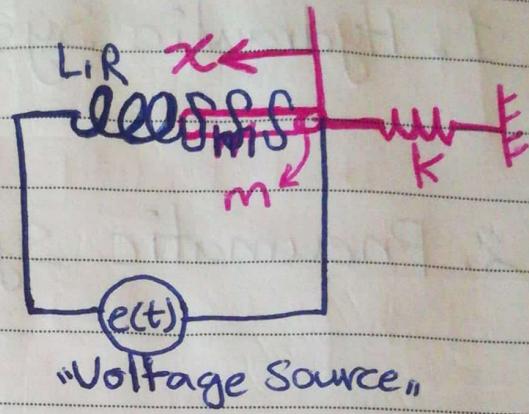
Coupling  $\rightarrow$   $\ddot{x} = \frac{Q}{\epsilon A}$   
is  $\ddot{x}$   $\rightarrow$   $\ddot{x}$  is  $\ddot{Q}$

## 5] Inductive Coupling

### Ex: Solenoid

is a linear system  
x and Q

because of displacement  
but it's not dependent  
with Q  $\rightarrow$  2 DOF



$$\text{Coil} = L + R$$

$$L = T - V$$

$$= \left[ \frac{1}{2} m \dot{x}^2 + \frac{1}{2} L \dot{Q}^2 \right] - \left[ \frac{1}{2} k x^2 \right]$$

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}} \right) - \frac{\partial L}{\partial x} = \phi$$

$$m \ddot{x} + kx = \phi \quad \dots \textcircled{1} \quad \text{for Mechanical}$$

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{Q}} \right) - \frac{\partial L}{\partial Q} = e(t) - R \dot{Q}$$

$$L \ddot{Q} + \phi = e(t) - R \dot{Q}$$

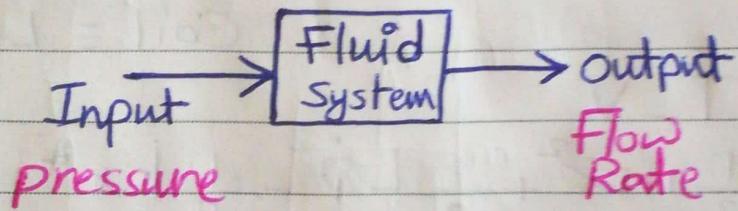
$$L \ddot{Q} + R \dot{Q} = e(t) \quad \dots \textcircled{2} \quad \text{for Electrical}$$

$$L \ddot{Q} + R \dot{Q} = e(t) \quad \dots \textcircled{2}$$

## Ch.5 : Fluid System

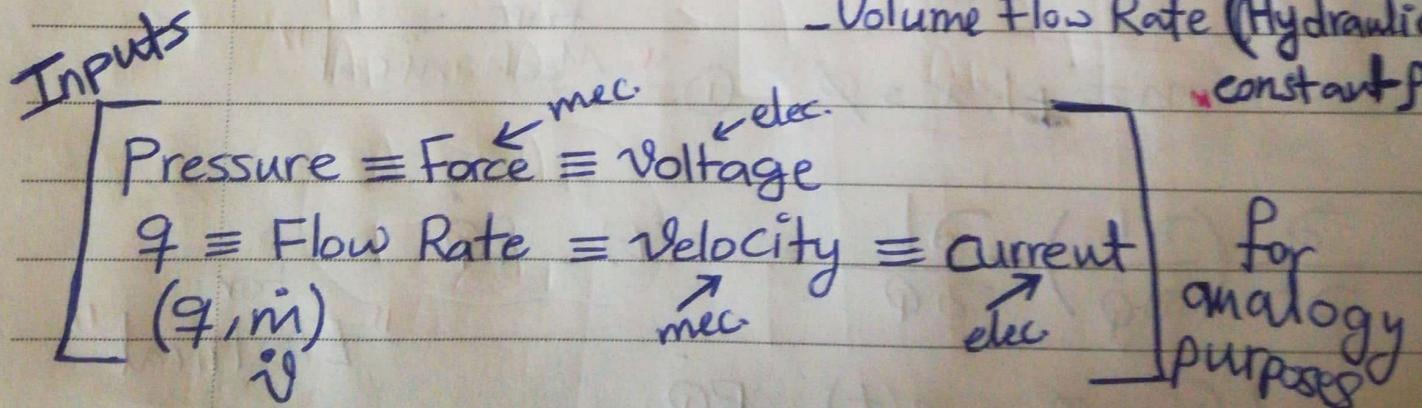
1. Hydraulic System  $\rightarrow$  Liquid (incompressible)  
 $P$  : constant

2. Pneumatic System  $\rightarrow$  gases (compressible)  
 $P$  : variable



- Mass Flow Rate (pneumatic)  
 Variables

- Volume Flow Rate (Hydraulic)  
 constant  $P_1$



# Fundamental Laws :- (Hydraulic System)

- ① Conservation of mass (Continuity Equation)
- ② Hydrostatic Pressure Law (Pascal Law)

\* # Conservation of mass :

$$\frac{\partial m}{\partial t} = \dot{m}_{in} - \dot{m}_{out} = 0 \rightarrow \dot{m}_{in} = \dot{m}_{out}$$

↓  
mass flow rate

[mass = density \* Volume]

$$\dot{V}_{in} = \dot{V}_{out}$$

volume flow Rate

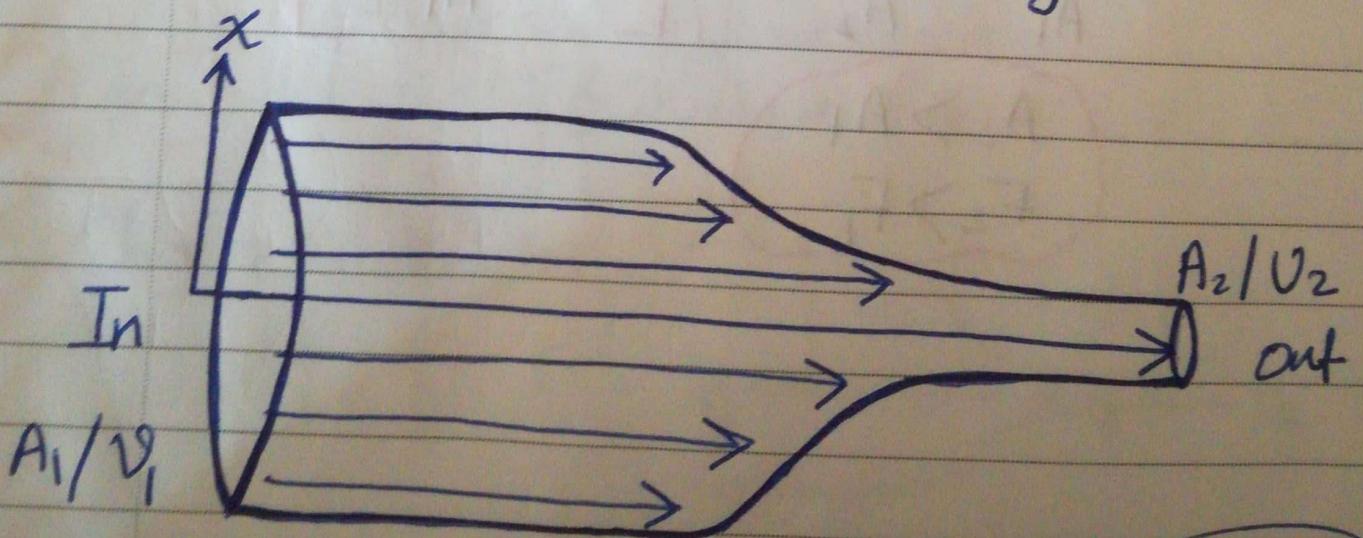
[Volume = Area \* Height]

$$A_1 \frac{\partial x_1}{\partial t} = A_2 \frac{\partial x_2}{\partial t}$$

$$A_1 V_1 = A_2 V_2$$

Velocity

الارتفاع متغير  
الزمن ثابت  
(A) ثابت



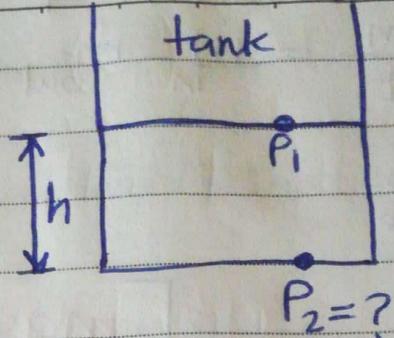
$t$  = volume  
 $v$  = velocity

# Hydrostatic Pressure Law (Pascal Law)  $\rightarrow$

\*  $P_1 = P_{atm}$

$$P_2 = P_{atm} + \rho gh$$

إذا كان الماء متساوياً

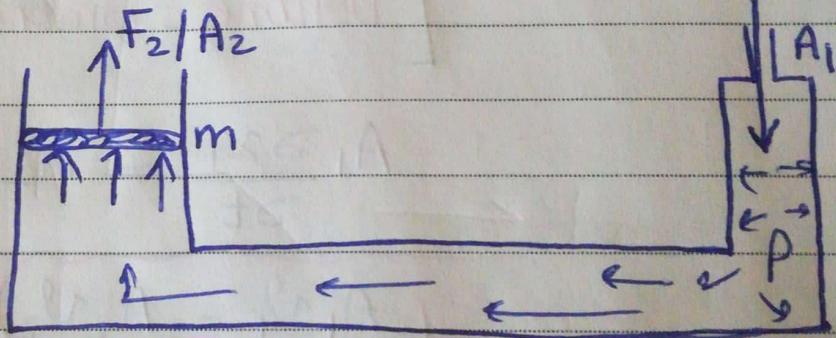


$$P_2 = \rho gh$$

بِنْيَانَ الْوَكَانِ مُعْلَقٌ مِّنْ أَعْلَى

„Pascal Law“

\*  $P = \frac{F}{A}$



$$\frac{F_1}{A_1} = \frac{F_2}{A_2} \rightarrow \left[ F_2 = \frac{A_2}{A_1} F_1 \right]$$

$A_2 > A_1$

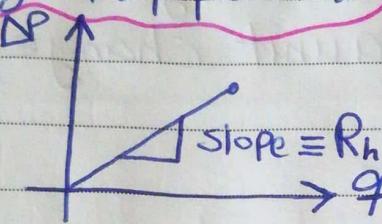
$F_2 > F_1$

# # The Basic Element modelling :-

## ① Hydraulic Resistance ( $R_h$ ) →

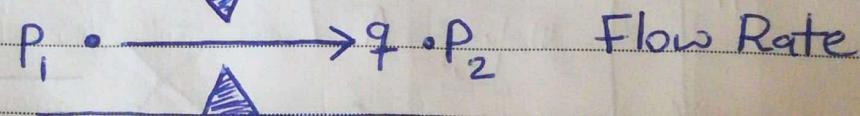
for  
hydraulic  
system

the resistance to flow because of valve or  
change in pipe diameter.  $\Delta P = R_h q$



damper  
ریز لوله

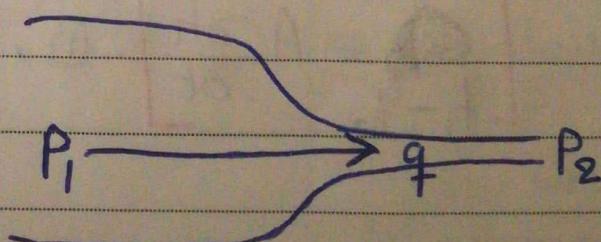
Orifice,



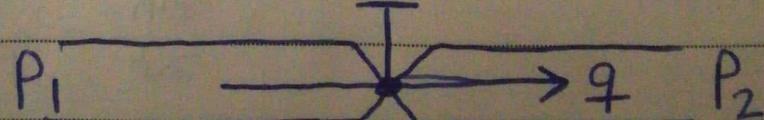
area change -  
میزان

resistance value in case -  
[Δ Area]  $\Rightarrow$  لوله میزان در این ایز

Change in diameter,



Valve,



$$P_1 \neq P_2$$

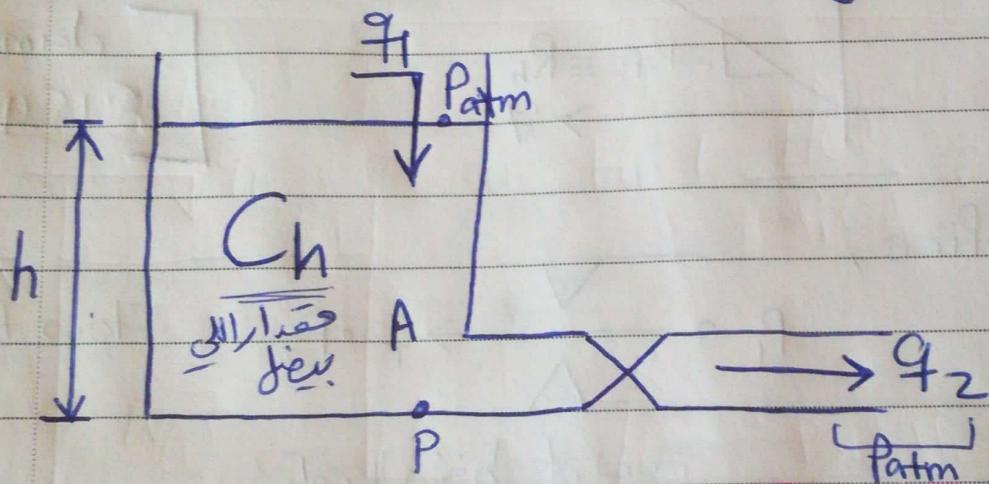
$$R_h = \frac{P_2 - P_1}{q}$$

FIVE APPLE

## ② Hydraulic Capacitance ( $C_h$ ) →

Used to describe energy storage with a liquid, where it's stored in the form of potential energy.

A height of liquid container is one form of storage, is the change in volume stored to cause a unit change in  $h$



$$q_1 - q_2 = \frac{\partial V}{\partial t}$$

$$q_1 - q_2 = \frac{\partial h}{\partial t} * A$$

Volume =  $Ah$   
↑ : Volume

$$q_1 - q_2 = A \frac{\partial h}{\partial t} \quad \text{... ①}$$

$$P = P g h \quad \text{... ②}$$

$$\frac{\partial P}{\partial t} = P g \frac{\partial h}{\partial t}$$

$$\frac{\partial h}{\partial t} = \frac{1}{Pg} \frac{\partial P}{\partial t} \quad \text{sub in ①}$$

$$\textcircled{1} \quad [q_1 - q_2] = \frac{A}{\rho g} \frac{\partial P}{\partial t}$$

$C_h$   $\rightarrow$  قدرة مضارع في الماء  
 قدرة مضارع في الماء  
 $Ah$   $\rightarrow$  ارتفاع كثافة الماء

$$E_c = \frac{1}{2} C_h \dot{V}^2$$

### ③ Hydraulic Inertance $\rightarrow$ Inductor II csj

It's equivalent to inertia/mass in mechanical system, and to the inductor in electrical system.

Long Pipe

$$F_1 - F_2 = P_1 A - P_2 A \quad \textcircled{1}$$

$$F_1 - F_2 = m a \rightarrow F_1 - F_2 = m \frac{\partial V}{\partial t} \quad \text{velocity} \quad \textcircled{2}$$

$$\textcircled{1} \quad \underbrace{A(P_1 - P_2)}_{\textcircled{2}} = \underbrace{ALP \frac{\partial V}{\partial t}}_{\textcircled{3}}$$

$$P_1 - P_2 = \frac{1}{A} \frac{\partial q}{\partial t}$$

$I_h$

mass = volume  $\times$   $\rho$

$m = \text{Area} \times \text{height} \times \rho$

$$m = ALP$$

flow rate

$q = \text{Area} \times \text{velocity}$

$$q = A \dot{V}$$

$$\frac{\partial q}{\partial t} = A \frac{\partial \dot{V}}{\partial t} \rightarrow \frac{\partial \dot{V}}{\partial t} = \frac{1}{A} \frac{\partial q}{\partial t}$$

sub in

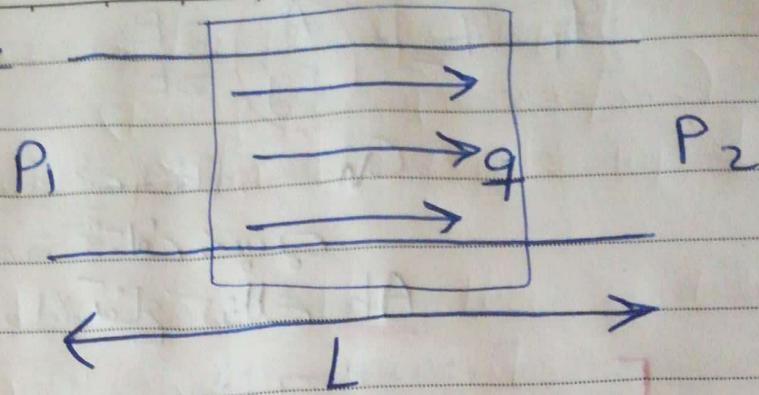
FIVE APPLE  $\textcircled{3}$

$$E_I = \frac{1}{2} L \dot{V}^2$$

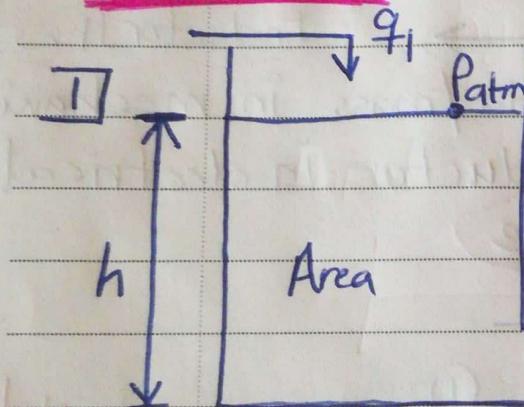
(Kinetic Energy) cijz

No.

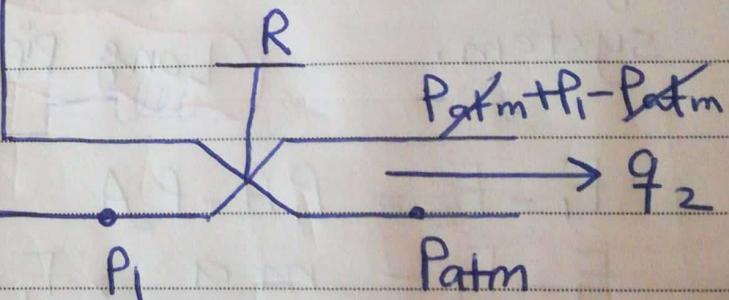
For Long pipe



## Examples



Find  $\frac{H(s)}{Q_1(s)}$  ? with zero initial conditions



where  $q$  : input flow rate

9<sub>2</sub> % Output " "

عند زياً في النظام من  $C_h$   $\leftarrow$  change in Area  $\leftarrow$

$$P_1 - P_2 = R q_2$$

$$P_1 - P_2 = R q_2$$

adj 109 11

Rw /  $\int g h(t) = R q_2$  !

~~Ch~~ 
$$q_1 - q_2 = A \frac{\partial h}{\partial t} \rightarrow q_1 = A \frac{\partial h}{\partial t} + \underline{q_2}$$

$$q_1 = A \frac{\partial h}{\partial t} + \frac{\rho g}{R} h(t)$$

~~as b~~

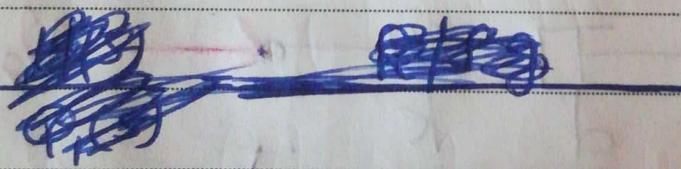
$$Q_1(s) = AS H(s) + \cancel{H(0)} + \frac{\rho g}{R} H(s)$$

$$Q_1(s) = H(s) \left[ \frac{R \cdot AS}{R} + \frac{\rho g}{R} \right]$$

$$\frac{H(s)}{Q_1(s)} = \frac{R}{\frac{(RA)}{\rho g} s + \frac{\rho g}{\rho g}} = \frac{R / \rho g}{(RC)s + 1}$$

$\downarrow$

$$Ch = \frac{A}{\rho g}$$



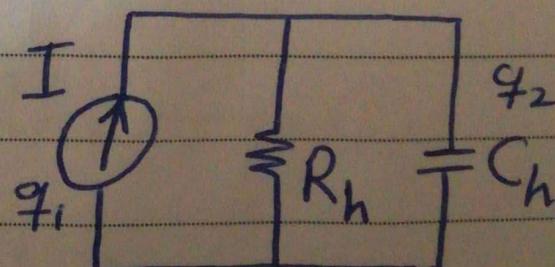
first Order system

→ the equivalent electrical system 80

$$Q_1(s) \approx I(s)$$

Area  $\approx$  Resistor

$q_2$  : Capacitor



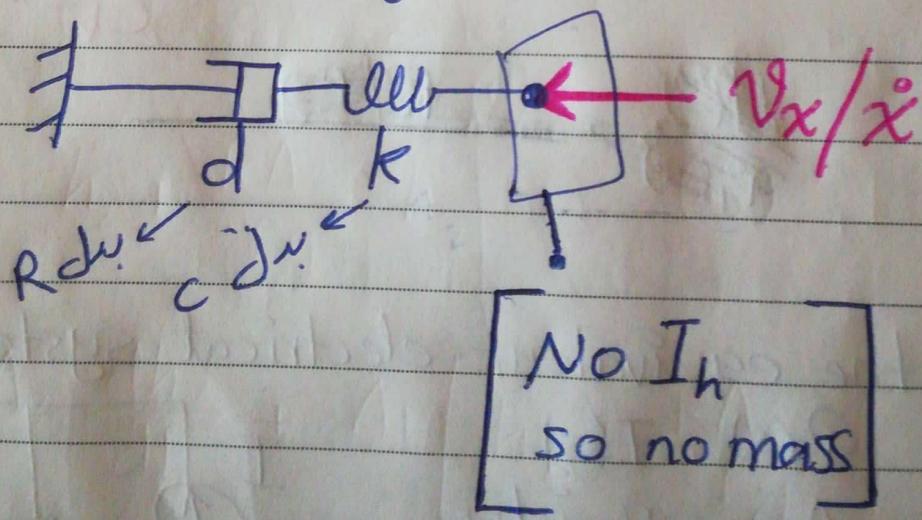
Voltage source  $\rightarrow$  for pressure input  
(pump / compressor / ...)

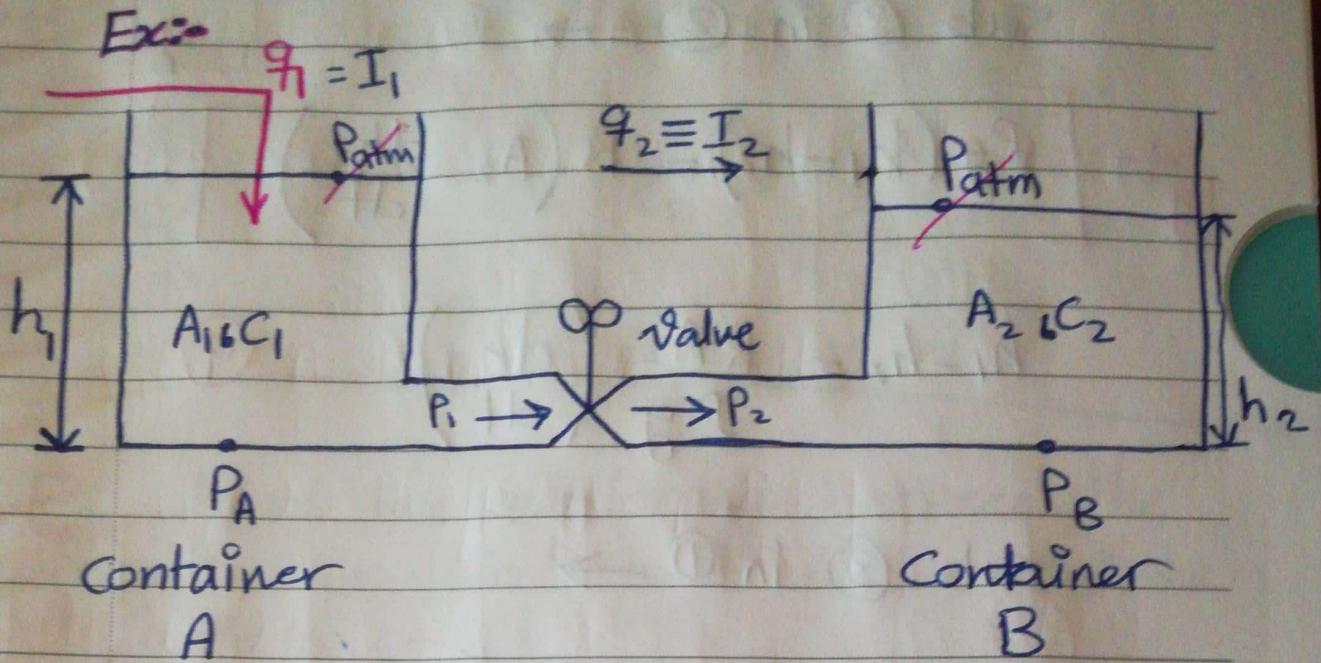
Current source  $\rightarrow$  for flow rate input

(Ch)  $\uparrow g_2$   $\downarrow$  Here, there are two  
(DA)  $\downarrow$  elements

$\rightarrow$  the equivalent mechanical system is  
hydraulic  $\leftrightarrow$  the  $\leftrightarrow$  mechanical

Input  $\equiv$  Velocity





# Find the differential equation that relates  $h_2$  to  $q_1$ ?

$$\left[ q_1 - q_2 = A_1 \frac{dh_1}{dt} \right] \quad \text{---} \quad ①$$

$$P_A - P_B = R q_z$$

$$Sg (h_1 - h_2) = Rg_2 \quad \text{--- (2)}$$

$$\left[ q_2 - \phi = A_2 \frac{dh_2}{dt} \right] \xrightarrow{\text{B جزء اول}} \text{B جزء اول} \quad (3)$$

ولو الى مخرج صنادقه  
بالاعتبار

\* ائمه انجلترا (اللي  
فأيّم R نفع  
حال

C<sub>2</sub> K

Sub ③ in ②  $\rightarrow$

$$\cancel{\rho g} \div \left[ \rho g (h_1 - h_2) = R \left( A_2 \frac{dh_2}{dt} \right) \right]$$

$$h_1 - h_2 = \frac{R A_2}{\rho g} \left( \frac{dh_2}{dt} \right) \quad \text{--- (4)}$$

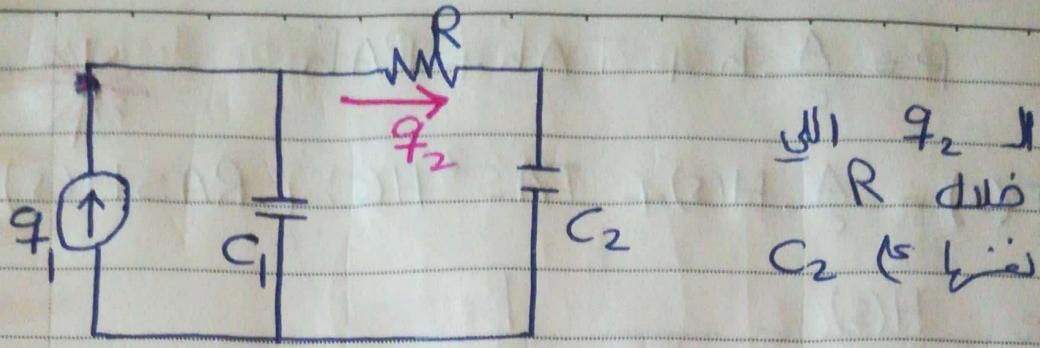
Sub ③ in ①  $\rightarrow$

$$q_1 = A_2 \frac{dh_2}{dt} + A_1 \frac{dh_1}{dt}$$

~~cancel~~ +

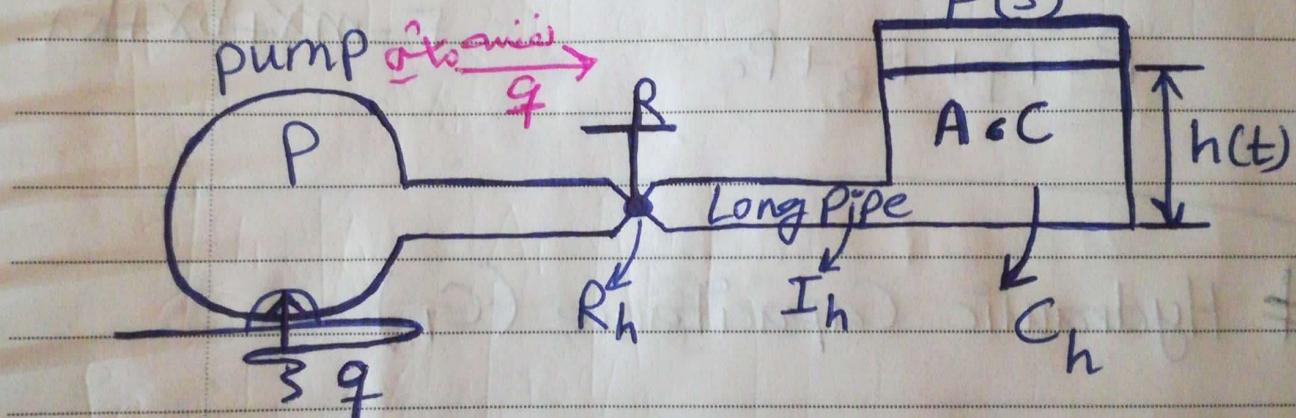
$$\dot{h}_1 = \dot{h}_2 + \frac{R A_2}{\rho g} \ddot{h}_2 \quad \text{--- (5)}$$

$$q_1 = A_2 \frac{dh_2}{dt} + A_1 \left[ \dot{h}_2 + \frac{R A_2}{\rho g} \ddot{h}_2 \right]$$



Ex Find the relationship between the input  $P$  required to fill the tank of the liquid.

$$\text{height } h(t) = \frac{H(s)}{P(s)}$$



$$q = q_c = q_I = q_c = Ah$$

$$P = P_g + P_I + P_R$$

$$P = \left( \frac{1}{C} \int q \right) + I \dot{q} + R q$$

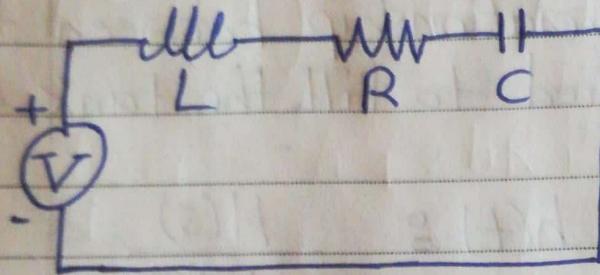
$$= \left( \frac{1}{C} \int Ah \cdot dt \right) + IA \ddot{h} + RA \dot{h}$$

$$P = \frac{A}{c} h + IA\ddot{h} + RA\dot{h}$$

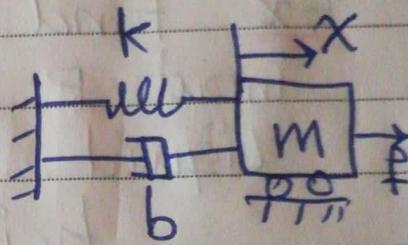


$$P(s) = \frac{A}{c} H(s) + IAS^2 H(s) + RASH(s)$$

$$\therefore \frac{H(s)}{P(s)} = \frac{1}{IAS^2 + RAS + \frac{A}{c}}$$



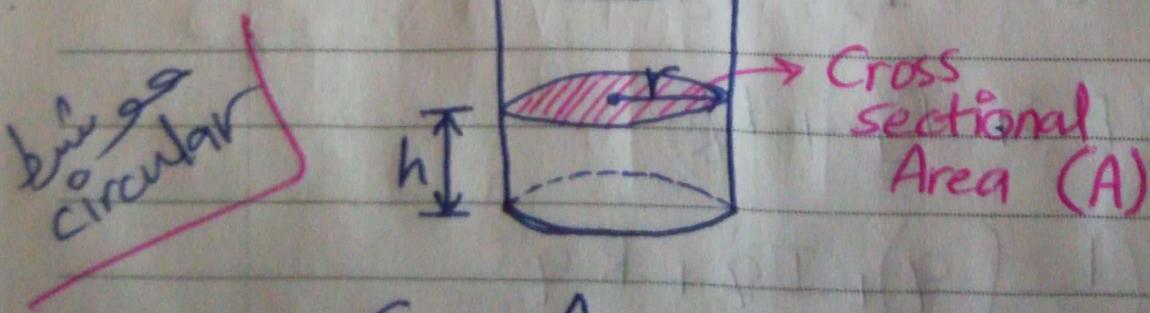
$$P = P_I + P_R + P_C$$



$$F = m\ddot{x} + b\dot{x} + kx$$

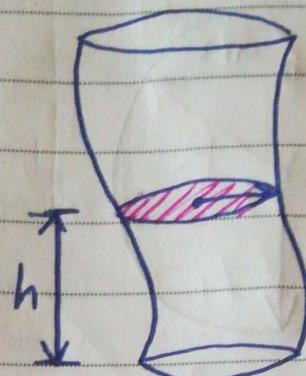
## # Hydraulic Capacitance ( $C_h$ )

### a. Vessel with constant Area



$$C_h = \frac{A}{\rho g}$$

b. Vessel with varying area

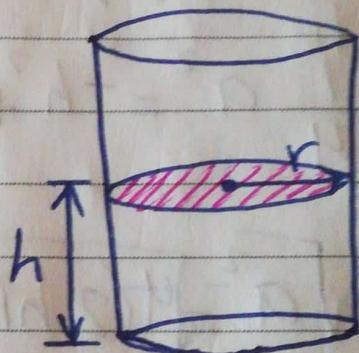


$$C_h = \frac{A(h)}{\rho g}$$

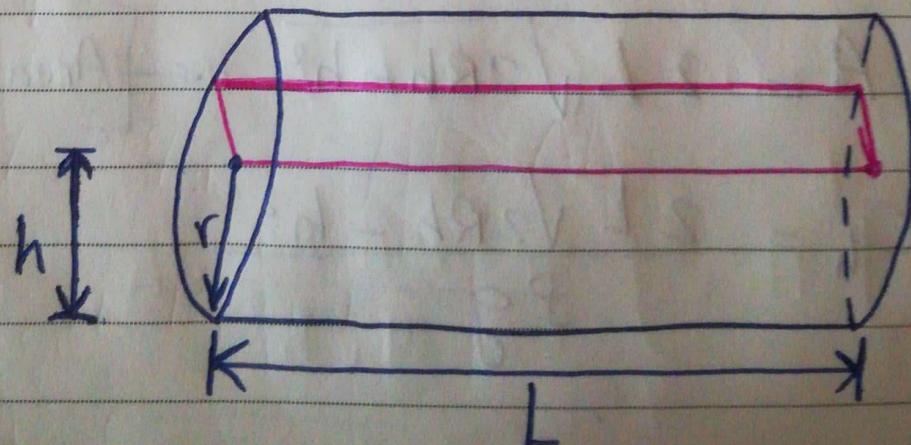
c. Vertical Cylindrical

$$C_h = \frac{A}{\rho g}$$

$$= \frac{\pi r^2}{\rho g}$$

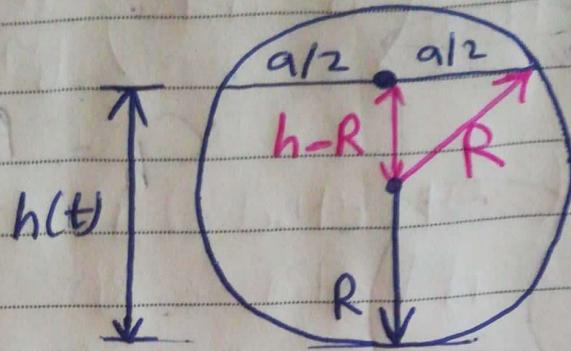


d. Horizontal Cylindrical



$$C_h = \frac{A(h)}{\rho g}$$

for the horizontal Cylindricals



$$R^2 = \left(\frac{a}{2}\right)^2 + (h-R)^2$$

$$\left[ \frac{1}{4} a^2 = -h^2 + 2hR + R^2 \Rightarrow R^2 \right] * 4$$

$$\sqrt{a^2} = \sqrt{4(2hR - h^2)}$$

$$a = 2 \sqrt{2hR - h^2}$$

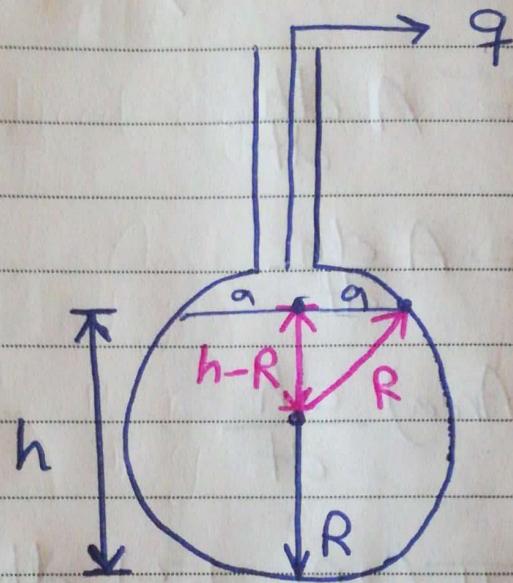
$$A = 2L \sqrt{2Rh - h^2} \quad \leftarrow \text{[Area} = L * a\text{]}$$

$$\therefore C_h = \frac{2L \sqrt{2Rh - h^2}}{\rho g}$$

Ex 8 For the spherical tank shown below :

1. prove that  $C_h = \frac{\pi}{\rho g} (2Rh - h^2)$

2. Determine how long it will be taken for the tank to empty, if the initial height is 9 m?



$$C_h = \frac{\text{Area}}{\rho g}$$

$$= \frac{\pi a^2 h}{\rho g}$$



$$a^2 = R^2 - (h-R)^2$$

$$a^2 = R^2 - h^2 + 2hR - R^2$$

$$a^2 = 2Rh - h^2$$

$$a = \sqrt{2Rh - h^2}$$

$$C_h = \frac{\pi}{\rho g} (2Rh - h^2) \quad \#$$

2) If  $A_o = 5.45 (10^{-3}) \text{ m}^2$   
 $V_o = 6.13 \sqrt{h}$   
 $R = 5 \text{ m}$       *Given*

$$q_{in} - q_{out} = A \frac{dh}{dt}$$

$$\phi - q_{out} = A \frac{dh}{dt}$$

$$-A_o V_o = (\text{Area}) \frac{dh}{dt}$$

$$-A_o V_o = \pi (2Rh - h^2) \frac{dh}{dt}$$

$$-5.45 (10^{-3}) (6.13 \sqrt{h}) = \pi (10h - h^2) \frac{dh}{dt}$$

$$\int_0^t dt = \int_{9 \text{ m}}^{0 \text{ m}} \frac{\pi (10h - h^2)}{-0.0334 \sqrt{h}} dh$$

$$t = \frac{-\pi}{0.0334} \left[ \frac{2}{3} (10h^{3/2}) - \frac{2}{5} h^{5/2} \right]$$

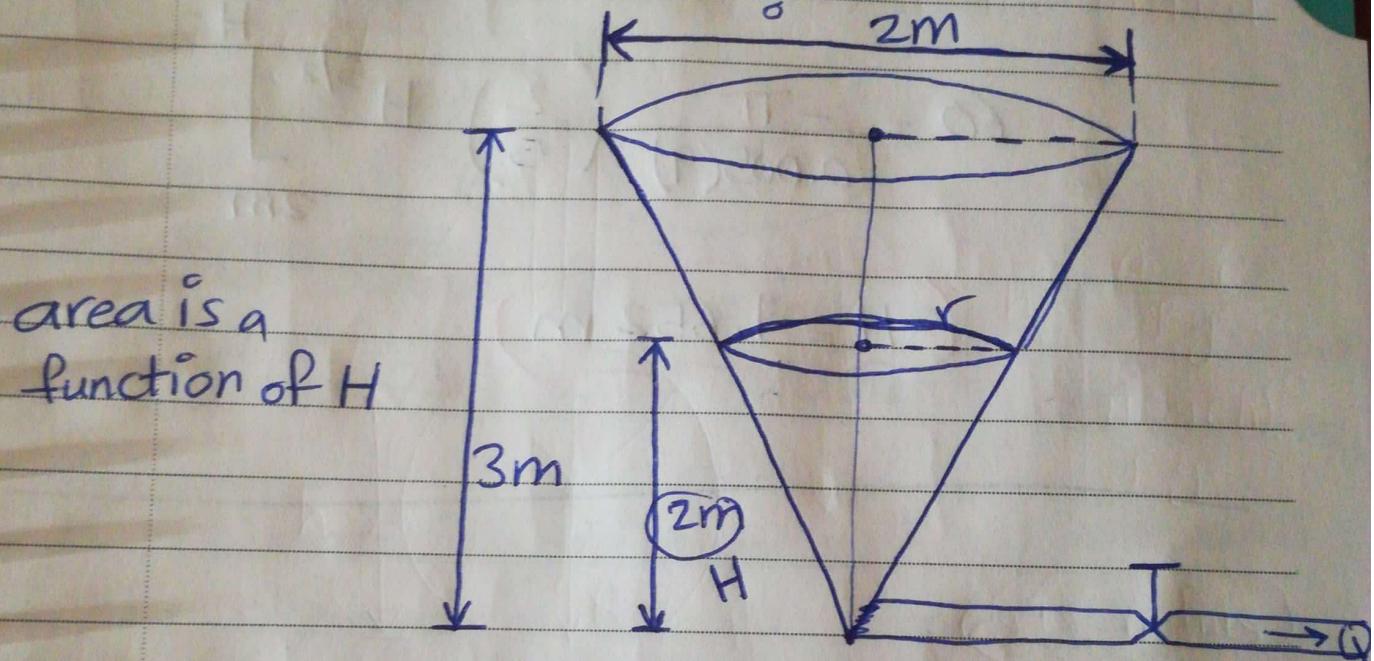
*t = 41 \text{ min}*      *#*

Ex Consider the conical water tank shown below:

$$Q = 0.005 \sqrt{H} \rightarrow \text{flow rate}$$

through the valve

the head is 2m at  $t=0$ , what will be the head at  $t=60$  s?



(Spherical Tank)  $\rightarrow$  if  $Q_{in} = Q_{out}$

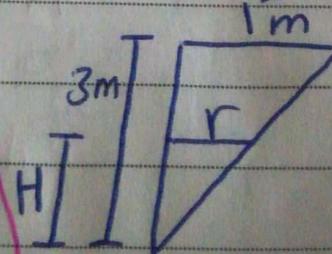
$$q_{in} - q_{out} = A \frac{dh}{dt}$$

$$\phi - 0.005 \sqrt{H} = A \frac{dH}{dt}$$

$Ch$   $\rightarrow$   $\frac{4}{3} \pi r^3$   
 if  $Q_{in} = Q_{out}$   $\rightarrow$   $\frac{4}{3} \pi r^3$   
 the volume of the tank is  $\frac{4}{3} \pi r^3$   
 if  $Q_{in} = Q_{out}$   $\rightarrow$   $\frac{4}{3} \pi r^3$

$$* A = \pi r^2$$

$$= \frac{\pi}{9} H^2$$



$$\frac{3}{H} \rightarrow \frac{1}{r}$$

$$r = \frac{H}{3}$$

FIVE APPLE

$$\frac{dt}{-0.005\sqrt{H}} \left( -0.005\sqrt{H} = \frac{\pi}{9} H^2 \frac{dH}{dt} \right)$$

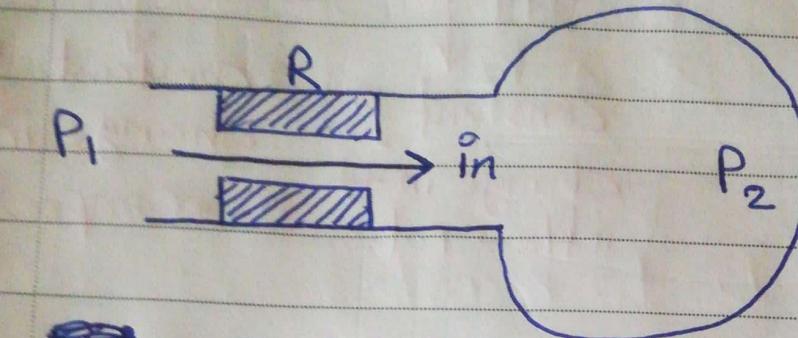
$$\int_6^{60} dt = \frac{-\pi}{9(0.005)} \int_{2m}^H H^2 * H^{-1/2} \circ dt$$

$$60 = \frac{-\pi}{0.005(9)} \left( \frac{2}{5} \right) H^{5/2} \Big|_{2m}^H$$

$$H = 1.652 \text{ m}$$

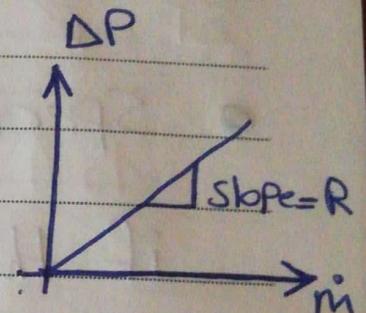
## No. # Pneumatic System →

Change in  $\Delta P$  required to cause a unit change in  $\dot{m}$  mass flow rate (in)



$P \equiv$  Voltage

$$R\dot{m} = P_1 - P_2 \Rightarrow \dot{m} \equiv \text{current}$$



①

\* Pneumatic Capacitance : in the mass  
the change in the gas stored (kg) required  
to cause a unit change in the gas pressure.

$$C_p = \frac{\partial m}{\partial P}$$

$$\dot{m} = \frac{\partial m}{\partial t} = \frac{\partial (P + \ddot{P})}{\partial t} \quad \begin{matrix} \leftarrow \\ \text{2 Variables} \\ \text{here...} \end{matrix}$$

$$\left( \dot{m} = P \frac{\partial \ddot{P}}{\partial t} + \ddot{P} \frac{\partial P}{\partial t} \right) \frac{\partial P}{\partial P}$$

$$\frac{\partial m}{\partial t} = \frac{\partial P}{\partial t} \left( P \frac{\partial T}{\partial P} + T \frac{\partial P}{\partial P} \right)$$

$$C_P = \frac{\partial m}{\partial P} = \left[ P \frac{\partial T}{\partial P} + T \frac{\partial P}{\partial P} \right]$$

↓  
Constant  $P$   
change in  $T$   
"flow rates"

↓  
constant  $T$   
change in  $P$   
"compressibility"

## • Special Cases 86

if the chamber is a fixed volume

$$\left[ C_P = T \frac{\partial P}{\partial P} \right]$$

for ideal gas and isothermal process

جایزه ای اینجا  
جزءی  
نظام

$$P = \rho R T$$

$$\frac{\partial P}{\partial P} = R T$$

$$\frac{\partial P}{\partial P} = \frac{1}{R T} \Rightarrow \left[ C_P = P \frac{\partial T}{\partial P} + T \frac{\partial P}{\partial P} \right]$$

② pneumatic inertance  $\rightarrow$ 

$$q = AV \quad \text{and} \quad \dot{m} = \rho q$$

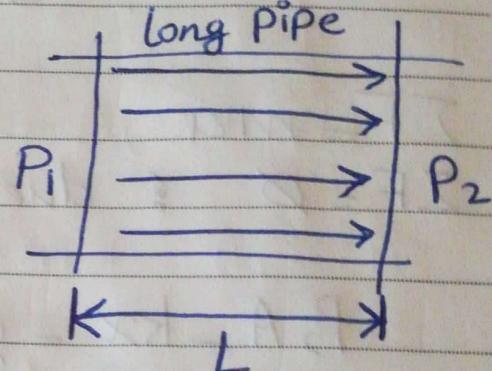
$$\text{pressure} = \frac{\text{Force}}{\text{area}}$$

$v$  is Velocity

$$\sum F = ma \rightarrow \dot{m} = \rho A v$$

$$(P_1 - P_2) A = m \frac{dv}{dt}$$

$$\left[ \Delta P = \frac{L}{A} \dot{m} \right]!$$



Ex) Find the I/O relationship

$$I = \text{input} = P_1$$

$$O = \text{output} = x$$

(Pneumatic system transfer func.)

It is applicable to  
piping & elements

It is also

Resistance  $\rightarrow$  Capacitance

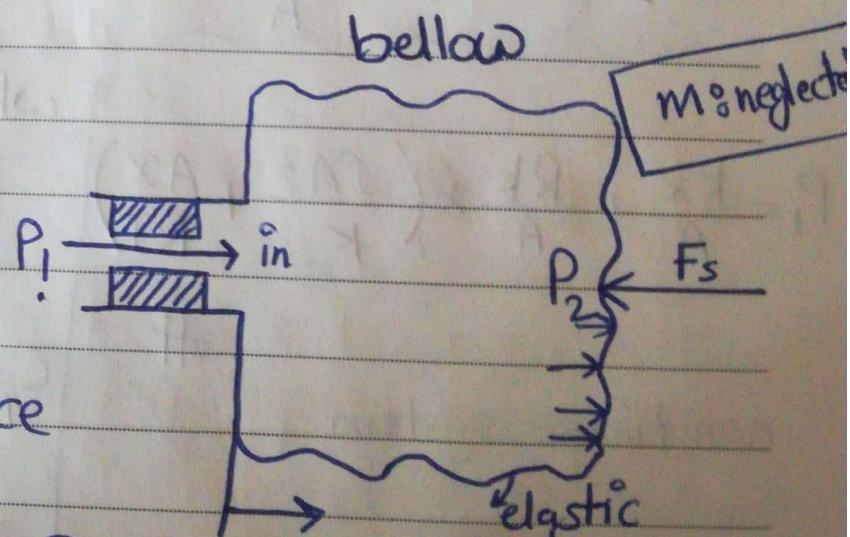
$$P_1 - P_2 = R \dot{m}$$

① Resistance Relation

$$\dot{m} = C_p(P_2)$$

② Capacitance Relation

pressure  
inside the bellow



$$(P_1 = P_2 + R \dot{m})$$

$$P_1 = P_2 + R C_p P_2 \quad (3)$$

$x \geq R \parallel$  السُّلُكُ مُعْطَى

$x \geq P_1$

$x \geq P_2$

$$\sum F = ma$$

$$\sum F = \emptyset \text{ due to massless}$$

$$P_2 A - kx = \emptyset \rightarrow \left[ P_2 = \frac{k}{A} x \right] \text{ sub in } (3)$$

$$P_2 = \frac{k}{A} \dot{x}$$

$$\Rightarrow (P_1 = \frac{k}{A} x + R C_p \frac{k}{A} \dot{x})$$

$$\text{where, } C_p = P \frac{\partial U}{\partial P_2} + T \frac{\partial P}{\partial P_2}$$

$$U = AX$$

$$P_1 = \frac{k}{A} x + \frac{Rk}{A} \dot{x} \left( \frac{PA^2}{K} + \frac{AX}{RT} \right)$$

#

non Linear system

$$C_p = P \frac{\partial (AX)}{\partial P_2} + AX \frac{\partial P}{\partial P_2}$$

$$= PA \frac{\partial X}{\partial P_2} + AX \frac{\partial P}{\partial P_2}$$

for ideal gas & isothermal process

$$C_p = \frac{PA^2}{K} + \frac{AX}{RT}$$

FIVE APPLE

(ideal gas)  $\rightarrow$   $\rightarrow$

$P_2 \parallel P \parallel$  السُّلُكُ مُعْطَى

الآن  $\rightarrow$   $\rightarrow$

الآن  $\rightarrow$   $\rightarrow$   $C_p$   $\rightarrow$  No info. \*

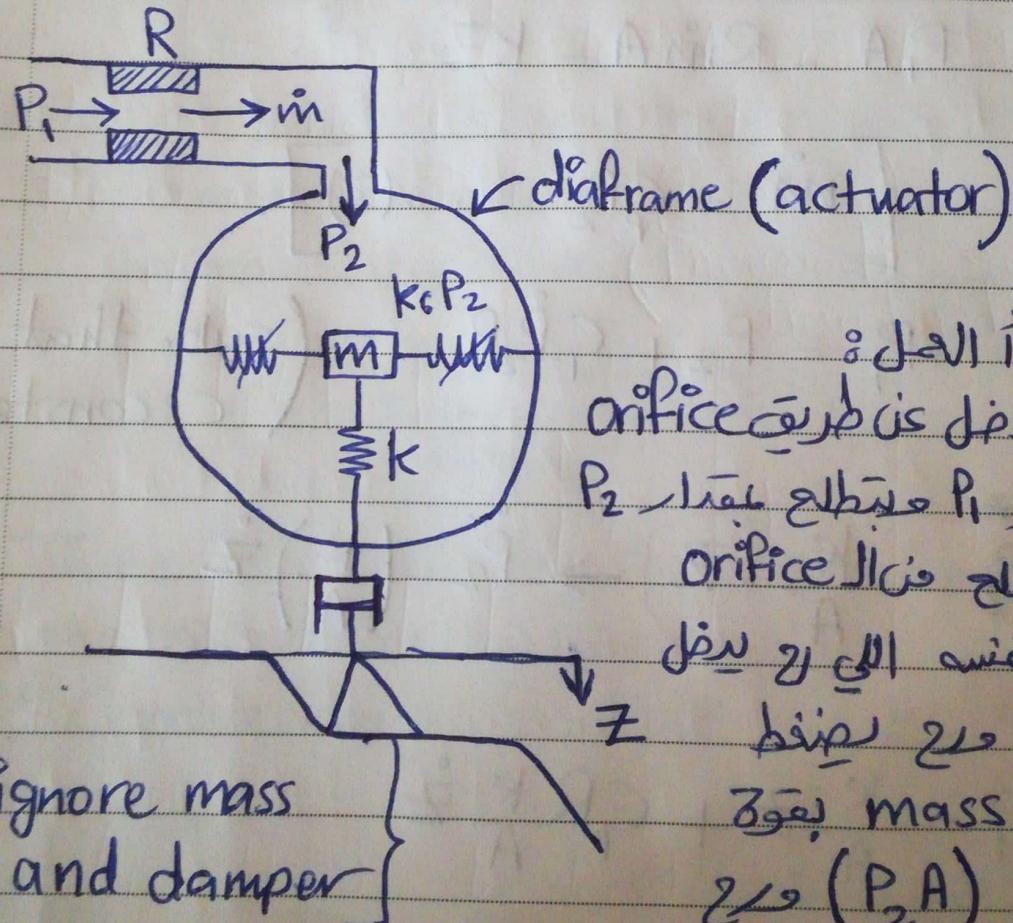
الآن  $\rightarrow$   $\rightarrow$   $(\bar{x})$   $\rightarrow$   $\rightarrow$

Where  $P_2 = \frac{kx}{A}$

١/٢P<sub>2</sub> يُبَيَّنُ وَيُقْسَمُ عَلَى

$$\frac{A}{K} \frac{\partial P_2}{\partial P_2} = \frac{K}{A} \left( \frac{\partial x}{\partial P_2} \right)$$

Ex Find  $\frac{Z(s)}{P_1(s)}$  ?



عند ادخاله من الاتصال  $P_1$  يخرج من الاورifice بسرعة  $V_1$  ويزداد ضغطه الى  $P_2$  فيكون
 
$$\frac{P_2}{P_1} = \frac{1}{1 + \frac{V_1^2}{2gD}}$$
 حيث  $D$  = نصف قطر الاورifice

$$\sum F = ma$$

$$P_2 A - KZ - b\ddot{Z} = ma$$

$$(P_1 - P_2) = R\dot{m} \quad \text{--- (1)}$$

$$P_2 = P_1 - R\dot{m} \quad \text{--- (2)}$$

$$(P_1 - R\dot{m})A - KZ - b\ddot{Z} = m\ddot{Z}$$

$$P_1 A - R\dot{m}A = KZ$$

$$\dot{m} = C_P \dot{P}_2 \quad \text{--- (3)}$$

$$P_1 = P_2 + C_P \dot{P}_2 R \quad \left( \begin{array}{l} \text{Sub. that} \\ C: \text{constant} \end{array} \right)$$

$$P_2 = \frac{K}{A} \cdot Z \rightarrow \dot{P}_2 = \left( \frac{K}{A} \right) \dot{Z}$$

$$\therefore P_1 = \frac{K}{A} Z + C_P \frac{K}{A} \dot{Z}$$

$$P_1(s) = \frac{K}{A} Z(s) + C_P \frac{K}{A} Z(s)$$

$$\therefore \frac{Z(s)}{P(s)} = \frac{\frac{A}{K}}{1 + C_P \frac{K}{A}}$$

السؤال السادس

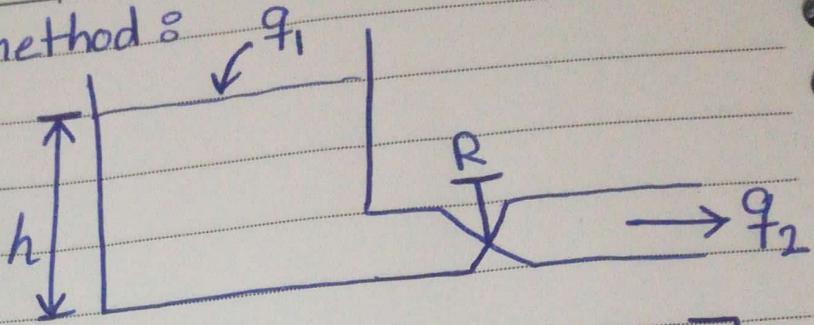
o: Element II is

Resistance  $\rightarrow$  valve

Inertance  $\rightarrow$  long pipe

Capacitance  $\rightarrow$  in

Q: Using Lagrange method  
Find  $\frac{H(s)}{Q_1(s)}$ ?



$$L = T - V$$

$$= \phi - \frac{1}{2C} \dot{h}^2$$

$$\frac{\partial}{\partial t} \left( \frac{\partial L}{\partial \dot{h}} \right) - \frac{\partial L}{\partial h} = F_{nc}$$

$$\phi + \frac{\dot{h}}{C} = -R q_2 = -R (q_1 + Ah)$$

pump  $\parallel$  tank  $\parallel$  outlet  
جهاز مضخة  $\parallel$  حوض  $\parallel$  صدر

$$q_1 - q_2 = Ah$$

$$q_2 = q_1 + Ah$$

$$\dot{h} = Ah$$

$$\frac{Ah}{RC} = -R \dot{q}_1 - \frac{RAh}{R}$$

الآن سنتغير معه  $\dot{h}$

$$\Rightarrow \dot{q}_1 = - \left( \frac{A}{RC} h + Ah \right)$$

$$Q_1(s) = - \left[ \frac{A}{RC} H(s) + ASH(s) \right]$$

عامل مشترك  $\rightarrow H(s)$   
وكل المعادلتين

$$\therefore \frac{H(s)}{Q_1(s)} = \frac{-1}{\frac{A}{RC} + \frac{RCA}{RC} s} = \frac{RC/A}{1 + (RC)s} = \frac{R/\rho g}{1 + (RC)s}$$

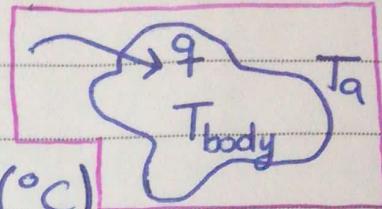
## ⇒ Thermal System

the transfer of heat from one substance to another.

$T$  = body temp. ( $^{\circ}\text{C}$ )

$T_a$  = ambient temp. ( $^{\circ}\text{C}$ )

$q$  = heat flow rate ( $\text{kcal/sec.}$ )



describing Variables ( $T, q$ )

→ Basic Building Elements :

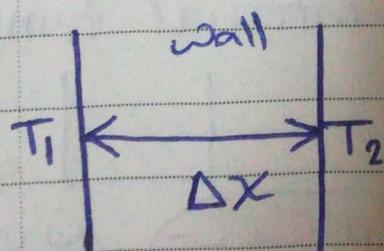
① Thermal Resistance ( $R_T$ )

② Thermal Capacitance ( $C_T$ ) [This is how it helps]

No Inertance in the thermal

→ Three different ways heat can flow from one substance to another :

① Conduction :  $q = k \Delta T$   $\leftarrow k = \frac{Ab}{\Delta x}$



$A$  // wall  $\downarrow$  wall's surface

$b$  // thermal conductivity  $\downarrow$  wall's  $\downarrow$   $b$

$\Delta x$  // wall thickness

② Convection  $q = k \Delta T \propto k = hA$

fluid  $\downarrow$   $\rightarrow$   
d $\downarrow$   $\rightarrow$   $\rightarrow$   $\rightarrow$

A // area

$h$  // convection  
connection  
fluid  $\downarrow$   $\rightarrow$   $\rightarrow$

③ Radiation

# Basic building elements  $\rightarrow$

1)  $R_T$  : property of material that indicates its ability to transfer heat.

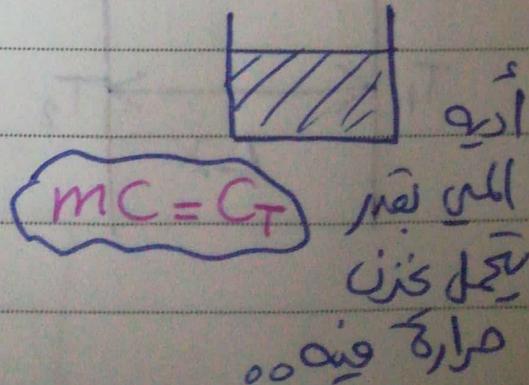
$$R_T = \frac{1}{k} = \frac{1}{Ah} \text{ "convection,"}$$

$$R_T = \frac{1}{k} = \frac{\Delta X}{Ab} \text{ "Conduction,"}$$

2)  $C_T$  : is the measure of heat energy required to increase the temp. of an object by a certain internal temp.

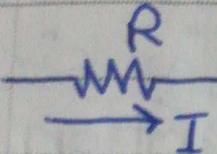
$$\rightarrow q = \cancel{mc} \frac{dT}{dt}$$

$$q = C_T \frac{dT}{dt}$$

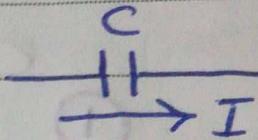


$m$  // mass of the substance  
 $C$  // Specific heat of substance  
 $q$  // rate of heat stored

# Thermal Analogy  $\rightarrow$



$$I = \frac{U}{R}, \quad q = \frac{\Delta T}{R_T}$$



$$I = C \frac{dU_c}{dt}$$

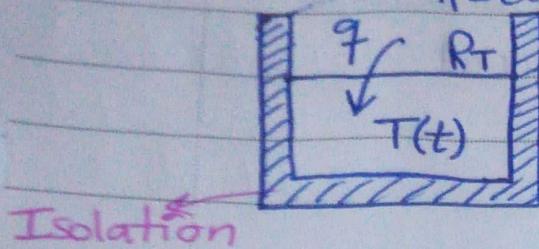
$$q = C_T \frac{dT}{dt}$$

$$\left\{ \begin{array}{l} i \equiv q \\ U \equiv \Delta T \\ R \equiv R_T \\ C \equiv C_T \end{array} \right.$$

[Note:  $\delta$  and  $\hat{\delta}$  are also used in the notes]

Ex1 Find  $T_1$  ?

$T_1 = \text{constant}$



استعمال اكوارد من برا جوا ←  
فلات الماء ت ←

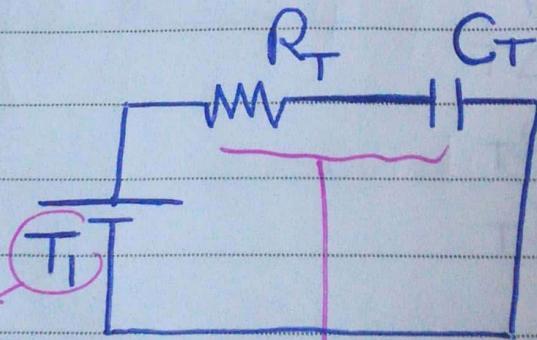
① Conduction

أعا فلات الـ ت ←  
بنـ ت ←

② Convection

$$q = \frac{T_1 - T(t)}{R_T} = C_T \frac{dT}{dt} \quad ②$$

$$T_1 = T(t) + R_T C_T \frac{dT}{dt}$$



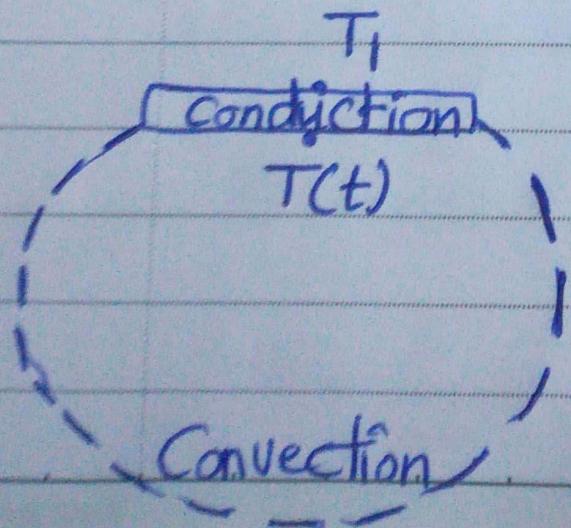
لأنه أنا حادفته كل رأة

وسبيـت q

{ هو فرقاـ لـ T بـ سـ يـار  
[V] اـذـة }

q Series  
بـ اـنـه فـقـمـاـ لـ T  
قطـاـ وـ تـوزـعـ عـلـيـهـ

نـقـيـلـ

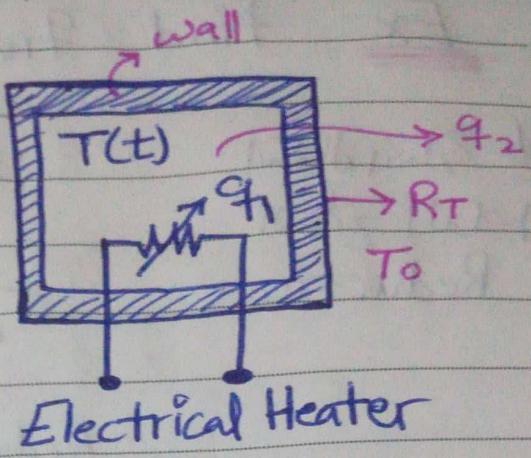


Ex.2 Find  $q_1$  ?

$$q_1 - q_2 = C_T \frac{dT}{dt} \rightarrow \text{Aug } T$$

$$q_1 = q_2 + C_T \frac{dT}{dt}$$

#  $q_2 = \frac{T(t) - T_0}{R_T} \quad \text{①}$

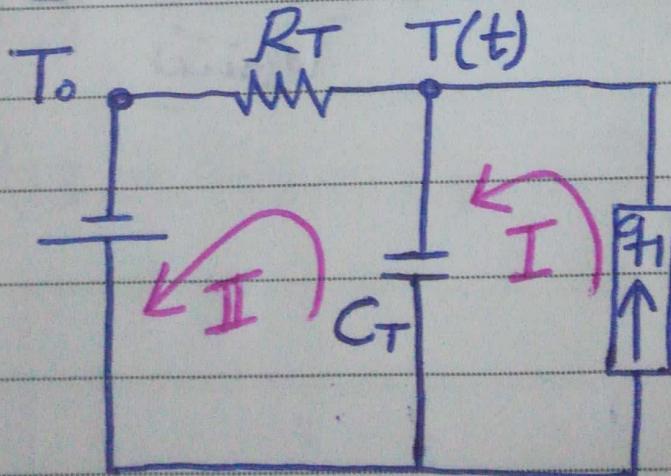
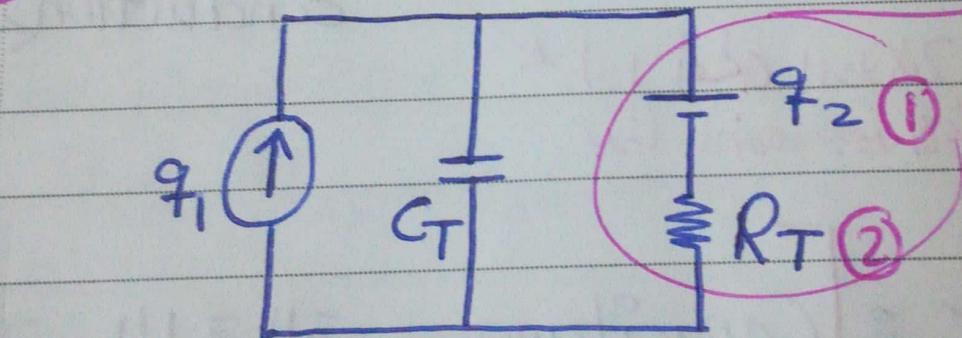


o In terms of temp.

$$q_1 = \frac{T(t) - T_0}{R_T} + C_T \frac{dT}{dt}$$

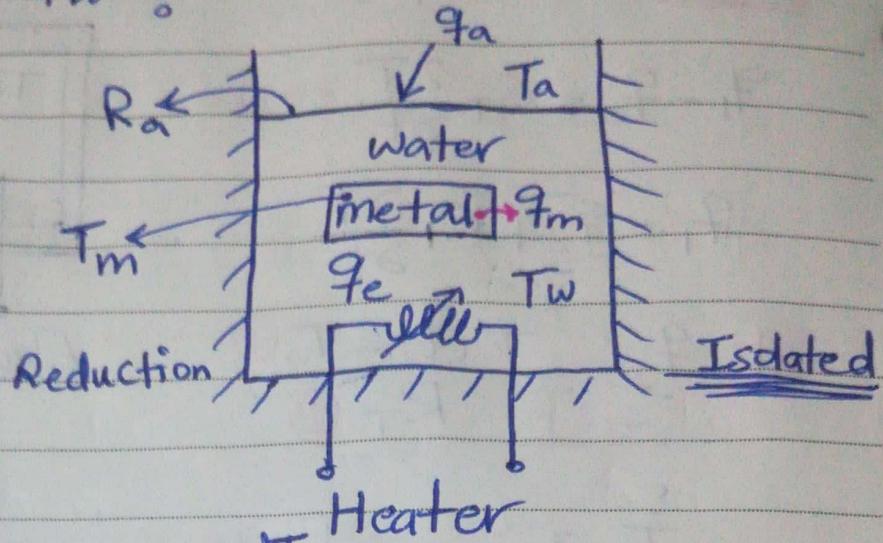
current

Voltage



Ex Find  $q_w$  ?

البيانات المطلوبة  
لحل المثلث  
Reduction



$$q_m + q_a + q_e = C_w \frac{\partial T_w}{\partial t}$$

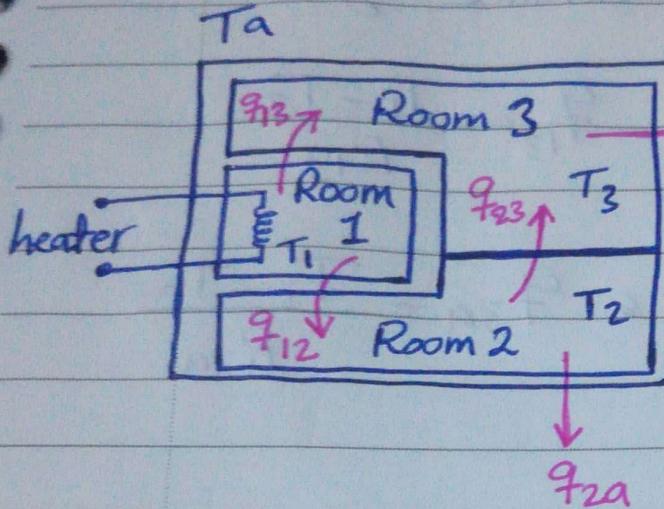
$$\left( \frac{T_m - T_w}{R_m} \right) + \left( \frac{T_a - T_w}{R_a} \right) + q_e = C_w \frac{\partial T_w}{\partial t}$$

conduction

إذا أمعن النظر  
في المثلث

Remark : Capacitance =  $\frac{\text{الطاقة}}{\text{النفخ}} = \frac{\text{الطاقة}}{\text{النفخ}} = \frac{\text{الطاقة}}{\text{النفخ}}$

Ex For the given system  
Find  $T_a$ ,  $T_1$ ,  $T_2$ , and  $T_3$ ?



(eq)  $\rightarrow$   $Q_1 = Q_2 + Q_3$

الآن يمكننا إيجاد  $T_1, T_2, T_3$

\* For Room (2) :

$$q_{12} - q_{2a} - q_{23} = C_2 \dot{T}_2$$

\* For Room (3) :

$$q_{13} + q_{23} - q_{3a} = C_3 \dot{T}_3$$

⇒ By adding eq<sub>1</sub>, eq<sub>2</sub> ::

$$q_{13} + q_{12} - q_{2a} - q_{3a} = C_3 \dot{T}_3 + C_2 \dot{T}_2$$

← ئىلىكى جىئىچىڭ قىزى ئەلەن

"Wall Resistance" جىئىچى

$$q_{12} = \frac{T_1 - T_2}{R_{12}} \quad , \quad q_{13} = \frac{T_1 - T_3}{R_{13}}$$

$$q_{2a} = \frac{T_2 - T_a}{R_{2a}} \quad , \quad q_{3a} = \frac{T_3 - T_a}{R_{3a}}$$